

AD-A121 487

STRENGTH OF MECHANICALLY FASTENED COMPOSITE JOINTS(U)
MICHIGAN UNIV ANN ARBOR DEPT OF MECHANICAL ENGINEERING
AND APPLIED MECHANICS F CHANG ET AL JUL 82

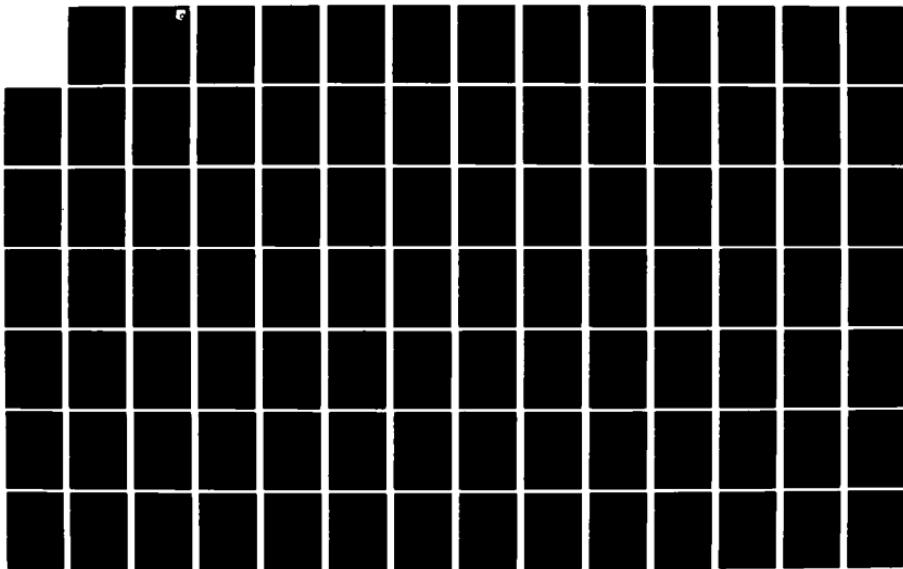
1/2

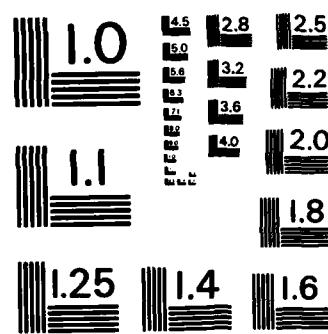
UNCLASSIFIED

AFWAL-TR-82-4095 F33615-81-C-5050

F/G 13/5

NL

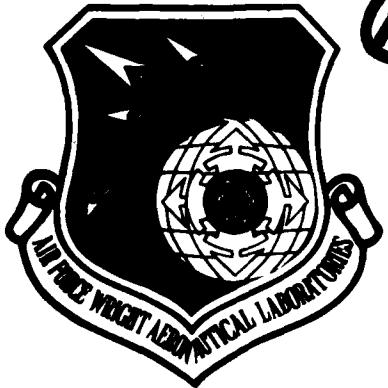




MICROCOPY RESOLUTION TEST CHART
NATIONAL BUREAU OF STANDARDS - 1963 - A

AD A 121 407

AFWAL-TR-82-4095



STRENGTH OF MECHANICALLY FASTENED COMPOSITE JOINTS

Fu-kuo Chang
Richard A. Scott
George S. Springer

Department of Mechanical Engineering and Applied Mechanics
The University of Michigan
Ann Arbor, MI 48109

July, 1982

Final Report for Period June 1981-May 1982

Approved for Public Release; Distribution Unlimited

MATERIALS LABORATORY
AIR FORCE WRIGHT AERONAUTICAL LABORATORIES
AIR FORCE SYSTEMS COMMAND
WRIGHT-PATTERSON AFB, OHIO 45433

DTIC
SELECTED
NOV 08 1982
S E D
E

82 11 08 048

NOTICE

When Government drawings, specifications, or other data are used for any purpose other than in connection with a definitely related Government procurement operation, the United States Government thereby incurs no responsibility nor any obligation whatsoever; and the fact that the Government may have formulated, furnished, or in any way supplied the said drawings, specifications, or other data, is not to be regarded by implication or otherwise as in any manner licensing the holder or any other person or corporation, or conveying any rights or permission to manufacture, use, or sell any patented invention that may in any way be related thereto.

This report has been reviewed by the Office of Public Affairs (ASD/PA) and is releasable to the National Technical Information Service (NTIS). At NTIS, it will be available to the general public, including foreign nations.

This technical report has been reviewed and is approved for publication.



S. W. TSAI, Project Engineer & Chief
Mechanics and Surface Interactions Branch
Nonmetallic Materials Division

FOR THE COMMANDER



F. D. CHERRY, Chief
Nonmetallic Materials Division

"If your address has changed, if you wish to be removed from our mailing list, or if the addressee is no longer employed by your organization please notify AFWAL/MLBM, W-PAFB, Ohio 45433 to help us maintain a current mailing list.

Copies of this report should not be returned unless return is required by security considerations, contractual obligations, or notice on a specific document.

Unclassified

SECURITY CLASSIFICATION OF THIS PAGE (When Data Entered)

REPORT DOCUMENTATION PAGE		READ INSTRUCTIONS BEFORE COMPLETING FORM
1. REPORT NUMBER AFWAL-TR-82-4095	2. GOVT ACCESSION NO. AD-H121407	3. RECIPIENT'S CATALOG NUMBER
4. TITLE (and Subtitle) STRENGTH OF MECHANICALLY FASTENED COMPOSITE JOINTS		5. TYPE OF REPORT & PERIOD COVERED June, 1981-May, 1982
		6. PERFORMING ORG. REPORT NUMBER
7. AUTHOR(s) Fu-kuo Chang Richard A. Scott George S. Springer		8. CONTRACT OR GRANT NUMBER(s) F33615-81-C-5050
9. PERFORMING ORGANIZATION NAME AND ADDRESS Department of Mechanical Engineering and Applied Mechanics, The University of Michigan Ann Arbor, Michigan 48109		10. PROGRAM ELEMENT, PROJECT, TASK AREA & WORK UNIT NUMBERS FY1457-81-02013
11. CONTROLLING OFFICE NAME AND ADDRESS Materials Laboratory (AFWAL/MLBM) Air Force Wright Aeronautical Laboratories Wright-Patterson, AFB, OH 45433		12. REPORT DATE July 1982
		13. NUMBER OF PAGES 97
14. MONITORING AGENCY NAME & ADDRESS (if different from Controlling Office)		15. SECURITY CLASS. (of this report) Unclassified
		15a. DECLASSIFICATION/DOWNGRADING SCHEDULE
16. DISTRIBUTION STATEMENT (of this Report) Approved for public release, distribution unlimited		
17. DISTRIBUTION STATEMENT (of the abstract entered in Block 20, if different from Report)		
18. SUPPLEMENTARY NOTES		
19. KEY WORDS (Continue on reverse side if necessary and identify by block number) Composite Materials Failure Hypothesis Joints Bolted Joints		
20. ABSTRACT (Continue on reverse side if necessary and identify by block number) A method is presented for predicting the failure strength and failure mode of mechanically fastened joints made of fiber reinforced composite laminates. The method includes two steps. First, the stress distribution in the laminate is calculated by the use of a finite element method. Second, the failure load and the failure mode are predicted by means of a proposed failure hypothesis together with Yamada's failure criterion. A computer code was developed which can be used to calculate the maximum load and the mode of failure of joints involving laminates with different ply orientations,		

Unclassified

SECURITY CLASSIFICATION OF THIS PAGE(When Data Entered)

different material properties, and different geometries. Results generated by the present method were compared to data and to existing analytical and numerical solutions. The results of the present method were found to agree well with those reported previously. Parametric studies were also performed to evaluate the effects of joint geometry and ply orientation on the failure strength and on the failure mode. -

Unclassified

i 1

SECURITY CLASSIFICATION OF THIS PAGE(When Data Entered)

FOREWORD

This report was prepared by Fu-kuo Chang, Richard A. Scott, and George S. Springer, Department of Mechanical Engineering and Applied Mechanics, The University of Michigan for the Mechanics and Surface Interactions Branch (AFWAL/MLBM), Nonmetallic Materials Division, Materials Laboratory, Air Force Wright Aeronautical Laboratories, Wright-Patterson AFB, Ohio. The work was performed under Contract Number F 33615-81-C-5050, Project number FY1457-81-02013.

This report covers work accomplished during the period June, 1981-May 1982.

Accession For	
NTIS GRA&I	<input checked="" type="checkbox"/>
DTIC TAB	<input type="checkbox"/>
Unannounced	<input type="checkbox"/>
Justification	
By _____	
Distribution/ _____	
Availability Codes	
Dist	Avail and/or Special
A	



TABLE OF CONTENTS

Section	Page
I. INTRODUCTION	1
II. PROBLEM STATEMENT	3
III. STRESS ANALYSIS - GOVERNING EQUATIONS	7
IV. STRESS ANALYSIS - FINITE ELEMENT METHOD	11
V. PREDICTION OF FAILURE	17
1. Failure Criterion	17
2. Failure Hypothesis - Characteristic Curve	18
3. Solution Procedure	20
VI. NUMERICAL SOLUTION	23
VII. RESULTS AND DISCUSSIONS	25
1. Isotropic and Orthotropic Plates	25
2. Failure Strength and Failure Mode	31
3. Effects of Geometry and Ply Orientations	38
VIII. CONCLUDING REMARKS	43
REFERENCES	44
APPENDIX A The Transformed Reduced Stiffness Matrix \bar{Q}_{ij}^P	46
APPENDIX B Shape Function Used in the Finite Element Code	48
APPENDIX C Listing of the Computer Code "BOLT", and a Sample of Input and Output	50

LIST OF ILLUSTRATIONS

Figure	Page
1. Geometry of the problem	4
2. Illustration of the three basic failure modes	5
3. Configuration of an elastic laminate with a loaded hole	8
4. Configuration of a joint approximated in the finite element method	12
5. Grid used in the finite element method. Right hand figure is an enlarged view of the grid around the hole	14
6. Description of the characteristic curve	19
7. Location of failure ($e=1$) along the characteristic curve	22
8. The stress σ_2 along the x_1 -axis in an isotropic infinite plate containing a circular hole. Comparison of the present results with the theoretical results given by Timoshenko [19]. Parameters used in the numerical calculations: $\bar{\sigma}=1.64$ MPa, $D=2R=7.62$ mm, $W/D=14$, $E/D=14$, $L/D=28$	26
9. The stress σ_2 along the x_1 -axis in an isotropic plate of finite width containing a loaded hole. Comparison of the present results with the theoretical results given by De Jong [8]. Parameters used in the numerical calculations: $D=7.62$ mm, $W/D=5.0$, $E/D=4.0$, $L/D=14.0$	29
10. The stress σ_2 along the x_1 -axis in an orthotropic finite plate $[0/90]_s$ containing a circular hole. Comparison of the present results with the theoretical results obtained by Nuismer and Whitney [18]. Parameters used in the numerical calculations: Material: Graphite/Epoxy T300/5208, $E_1=149.8$ GPa, $E_2=11.2$ GPa, $G_{12}=5.39$ GPa, $v_{12}=0.29$, $\bar{\sigma}=2.3$ MPa, $D=24.5$ mm, $W/D=3.0$, $E/D=4.0$, $L/D=14.0$	30
11. The effects of width ratio on the failure load of laminates with different ply orientations. P_f is the tensile failure load of laminates without holes. Parameters used in the numerical calculations: Material: Graphite/Epoxy T300/SP286, $W=38$ mm, $E=50.8$ mm, $L=203.2$ mm, $H=1.067$ mm for $[0/\pm 45/90]_s$ and $[0_2/\pm 45]_s$, and $H=1.18$ mm for $[0/90]_{2s}$	39
12. The effects of edge ratio on the failure load of laminates with different ply orientations. Parameters used in the	

numerical calculations: Material: Graphite/Epoxy T300/SP286
D=5.08 mm, W/D=5, L/D=14

40

13. The effects of maximum ply angle ϕ and ply continuity $\Delta\theta$ on the failure load of mechanically fastened joints. Parameters used in the numerical calculations: Material: Graphite/Epoxy T300/SP286, D=4.76 mm, W/D=5.336, E/D=2.993, L/D=14.68
H=1.397 mm

41

14. Geometry of an element used in the finite element calculations. Left: Element in the x_1 - x_2 coordinate system. Right: Element (master element) in the local (r-s) coordinate system. $x_{i\alpha}$ is the coordinate of node α in the i direction, $q_{i\alpha}$ is the displacement of node α in the i direction and (r_α, s_α) are the coordinates of node α in the r-s coordinate system, $i=1, 2$, $\alpha=1, 2, 3$ or 4

49

LIST OF TABLES

Table	Page
1. Input parameters required by the computer code and the output provided by the code	24
2. Stress concentration factor (SCF) around a pin loaded hole contained in an isotropic plate of infinite width. Comparison of present results with those obtained by previous investigators	28
3. Summary of test conditions	32
4. Material properties used in the calculations	33
5. Comparisons between the experimental (P) and predicted (P_c) failure loads. Case numbers correspond to test conditions given in Table 3	34
6. Comparisons between the experimental failure loads and the values predicted by Waszczak and Cruse [1]	36
7. Comparisons of predicted failure modes with those observed experimentally. T-Tension Mode, S-Shearout Mode, B-Bearing Mode	37

LIST OF SYMBOLS

A	Total surface area of the laminate
A_L	Stress prescribed area
A_F	Stress free area
A_R	Displacement prescribed area (fixed boundary)
A_{Lg}	Surface area of an element g on which surface tractions is applied
B	Bearing stress
D	Diameter of the hole
E	Edge distance
E_{ijkl}	Elastic moduli
E_{mn}	Reduced laminate moduli
e	Failure indicator ($e < 1$ non-failure, $e \geq 1$ failure)
e_o	Maximum value of e on the characteristic curve
$F_{i\beta}$	Assembled load vector
H	Thickness of the laminate
h^p	Thickness of the p -th ply
$k_{i\beta ka}^g$	Stiffness matrix of the q -th element
$\bar{K}_{i\beta ka}$	Assembled stiffness matrix
L	Plate length
M	Number of elements
N	Number of plies in the laminate
N_α	Shape function
n_j	Unit vector normal to the surface
P	Applied Load
P_{max}	Failure (maximum) load
\tilde{Q}_{ij}^p	Transformed reduced stiffness matrix of the p -th ply

q_{ia}	Nodal displacement
r	Radial distance
r_c	Radial distance to the characteristic curve
R_{ot}	Characteristic length for tension
R_{oc}	Characteristic length for compression
s	Ply shear strength
s_c	Shear strength of cross-ply laminate
s	Total area of the two-dimensional laminate
s_g	Area of element g (2-dimensions)
SCF	Stress concentration factor
T_i	Surface traction components
u_i	Displacement
\bar{u}_i	Arbitrary displacement functions
v_o	Total volume of the laminate
v_g	Volume of element g
w	Width of the plate
x	Ply tensile strength
x	Coordinate along the fiber direction in each ply
x_1	Coordinate perpendicular to the loading direction in the laminate plane
x_2	Coordinate opposed to the loading direction and perpendicular to the x_1 -axis
x_3	Coordinate perpendicular to the x_1 and x_2 axes
y	Coordinate perpendicular to the fiber direction in each ply
Γ_L	Boundary curve of the hole on which the surface traction is applied
Γ_{Lg}	Boundary curve of the element g on which the surface traction is applied
$\Delta\theta$	Ply continuity

ϵ_{ij} Strain components in the x_1-x_2 coordinate system
 η Angle measured counterclockwise from the x_1 -axis
 θ_f Angle at which failure occurs
 σ_{ij} Stress components in the x_1-x_2 coordinate system
 $\sigma_x, \sigma_y, \sigma_{xy}$ Stress components in the $x-y$ coordinate system
 ϕ Maximum angular range of the ply orientation

SECTION I

INTRODUCTION

Among the major advantages of laminated composite structures over conventional metal structures are their comparatively high strength to weight and stiffness to weight ratios. As a result, fiber reinforced composite materials have been gaining wide application in aircraft and spacecraft construction. These applications require joining composites either to composites or to metals. Most commonly, joints are formed using mechanical fasteners. Therefore, suitable methods must be found to determine the failure strengths of mechanically fastened joints. A knowledge of the failure strength would help in selecting the appropriate size joint in a given application.

Owing to the significance of the problem, several investigators have developed analytical procedures for calculating the strength of bolted joints in composite materials. Among the recent studies are those of Waszczak and Cruse [1], Agarwal [2], and Garbo and Ogonowski [3]. As will be discussed in Section VII, the previous methods provide conservative results and underestimate the failure strength, often by as much as fifty percent.

The major objective of this investigation was, therefore, to develop a method which a) predicts the failure strength and failure mode of mechanically fastened composite joints with better accuracy than the existing analytical methods and b) can be used readily in the design of mechanically fastened composite joints. In the present method first the stress distribution around the hole is calculated by the use of a finite element method. Second, the failure load and the failure mode are predicted by means of a proposed new failure

hypothesis together with Yamada's [4] failure criterion. On the basis of this analysis a computer code was developed which can be applied to joints involving laminates with different ply orientations, different material properties, and different configurations, including different hole sizes, hole positions, and joint thicknesses. Because of the accuracy of the method and the flexibility of the computer code, the code can be applied to the analysis and the design of mechanically fastened composite joints.

SECTION II

PROBLEM STATEMENT

Consider a plate (length L , width W , thickness H) made of N fiber reinforced unidirectional plies. The ply orientation is arbitrary, but must be symmetric with respect to the $x_3=0$ plane (symmetric laminate, Figure 1). Perfect bonding between each ply is assumed.

A hole of diameter D is located along the centerline of the plate ($x_1=0$) at a distance E from one end of the plate. A rigid pin (diameter D), supported outside the laminate, is inserted into the hole (Figure 1). A uniform tensile load P is applied to the plate, as shown in Figure 1. The load is parallel to the plate (in-plane loading) and is symmetric with respect to the centerline. Hence the load cannot create bending moments about either the x_1 , x_2 or x_3 axes. Moreover, for symmetric laminates, in plane and bending effects are uncoupled. It is desired to find

- 1) the stresses and strains in each ply,
- 2) the maximum (failure) load (P_{\max}) that can be applied before the joint fails, and
- 3) the mode of failure.

Point 2 refers to the fact that, according to experimental evidence, mechanically fastened joints under tensile loads generally fail in three basic modes referred to as tension mode, shearout mode, and bearing mode. The type of damage resulting from each of these modes is illustrated in Figure 2. The objective, listed in point 3 above, is to determine which of these modes will most be responsible for the failure.

The calculation proceeds in three steps. For a given geometry and

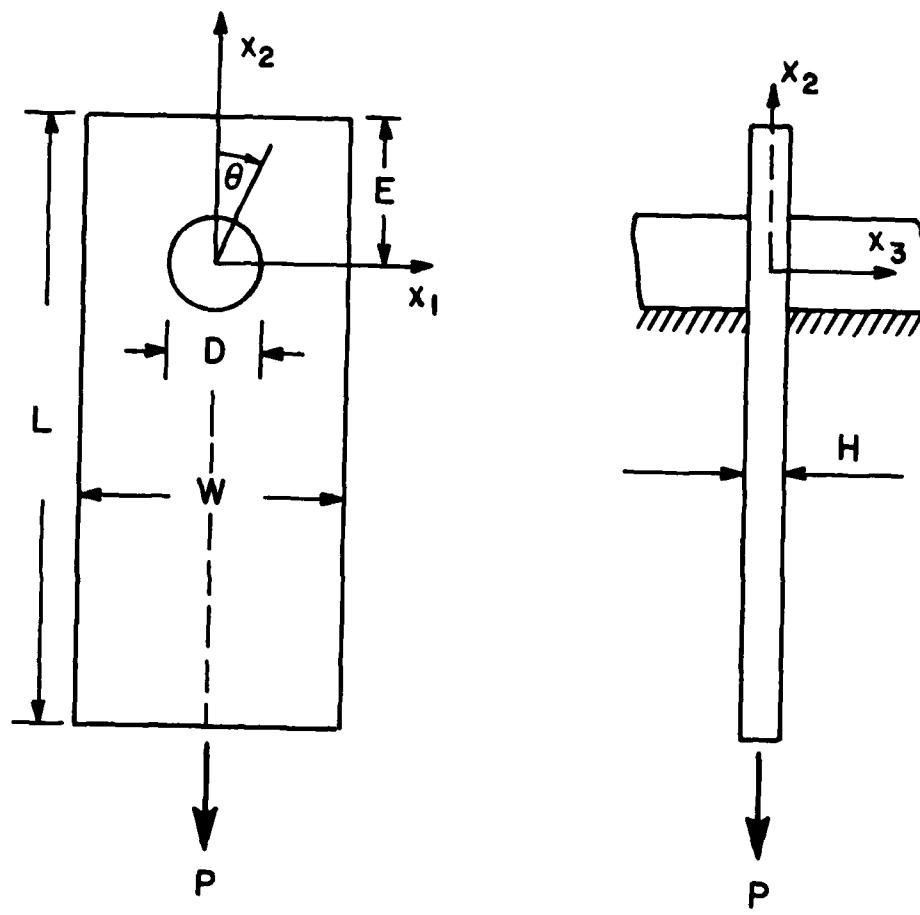


Figure 1. Geometry of the problem

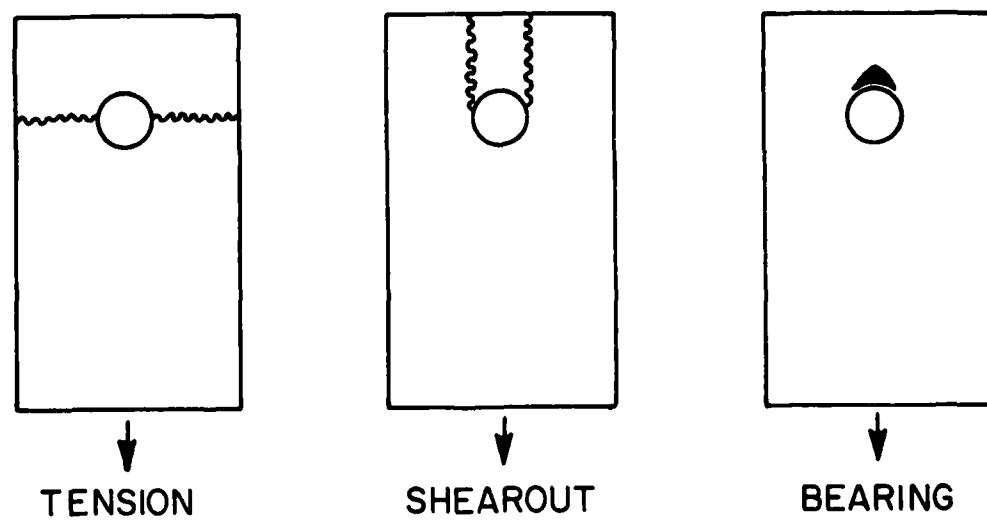


Figure 2. Illustration of the three basic failure modes

load

- 1) the stress distribution around the hole is calculated,
- 2) the maximum (failure) load is predicted , and
- 3) the mode of failure is determined.

The details of these steps are presented in Sections III-V.

SECTION III

STRESS ANALYSIS-GOVERNING EQUATIONS

The stresses in the laminate are calculated on the basis of anisotropic theory of elasticity and classical lamination plate theory. Accordingly, in the analysis planes are taken to remain planes, the strain across the thickness is taken to be constant [$\epsilon_{ij} = f(x_1, x_2)$] and only plane stresses are considered ($\sigma_{13} = \sigma_{23} = \sigma_{33} = 0$). Under these conditions, in the absence of body forces, the condition of force equilibrium can be expressed as [5]

$$\begin{aligned}\frac{\partial \sigma_{11}}{\partial x_1} + \frac{\partial \sigma_{12}}{\partial x_2} &= 0 \\ \frac{\partial \sigma_{21}}{\partial x_1} + \frac{\partial \sigma_{22}}{\partial x_2} &= 0\end{aligned}\tag{1}$$

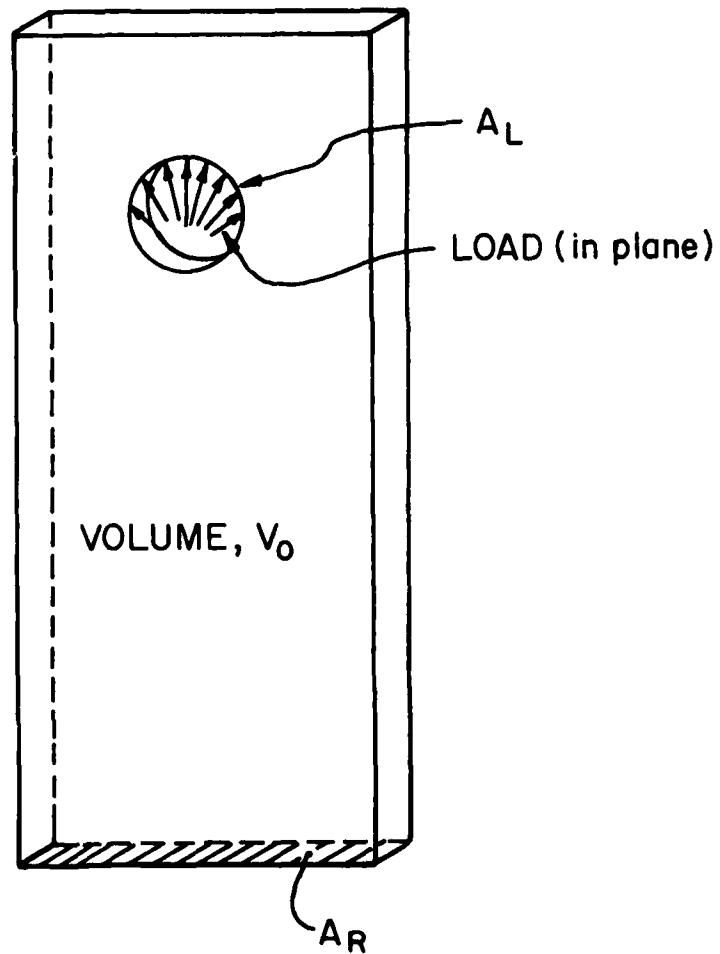
In index notation eq. (1) becomes

$$\sigma_{ij,j} = 0\tag{2}$$

σ_{ij} is the stress in the plane normal to the x_i axis and is in the x_j direction. The subscripts i and j may have the values 1 or 2. Consider now an elastic laminate of volume V_0 containing a loaded hole as shown in Figure 3. Stresses are applied over the surface area A_L . The surface area A_R is rigidly fixed (no displacement), while the surface area A_F is free of applied stress. The total surface area is

$$A = A_L + A_R + A_F\tag{3}$$

Let us denote by \bar{u}_i any arbitrary displacement inside the body. \bar{u}_i is a test function. The only requirement is that \bar{u}_i be continuous, differentiable and be zero on A_R . By multiplying eq. (2) by \bar{u}_i and by taking the volume integral of the



TOTAL SURFACE: A

LOADED SURFACE: A_L

FIXED SURFACE: A_R

STRESS FREE SURFACE: $A_F = A - A_L - A_R$

Figure 3. Configuration of an elastic laminate with a loaded hole

resulting expression we obtain

$$\iiint_{V_o} \sigma_{ij,j} \bar{u}_i \, dV = 0 \quad (4)$$

By employing the identity

$$\sigma_{ij,j} \bar{u}_i = (\sigma_{ij} \bar{u}_i)_j - \sigma_{ij} \bar{u}_{i,j} \quad (5)$$

and by utilizing Gauss' theorem, eq. (4) may be written as

$$\iint_A \sigma_{ij} n_j \bar{u}_i \, dA - \iiint_{V_o} \sigma_{ij} \bar{u}_{i,j} \, dV = 0 \quad (6)$$

where n_j is the unit vector normal to the surface. On the free surface A_F the stresses are zero, while on the surface A_R the displacement is zero. These conditions give

$$\iint_{A_F} \sigma_{ij} n_j \bar{u}_i \, dA = 0 \quad (7)$$

$$\iint_{A_R} \sigma_{ij} n_j \bar{u}_i \, dA = 0 \quad (8)$$

The force per unit area (called surface traction) at each point of the surface area A_L is [5]

$$T_i = \sigma_{ij} n_j \quad (9)$$

Equations (6) - (9) yield

$$\iint_{A_L} T_i \bar{u}_i \, dA = \iiint_{V_o} \sigma_{ij} \bar{u}_{i,j} \, dV \quad (10)$$

The stress is related to the displacement through the stress-strain relationship, which for an elastic body is

$$\sigma_{ij} = E_{ijkl} \epsilon_{kl} \quad (11)$$

The subscripts k and l may take on the values of 1 or 2. The strains are related to the displacements u_i by the expression

$$\epsilon_{kl} = \frac{1}{2} \left(\frac{\partial u_k}{\partial x_l} + \frac{\partial u_l}{\partial x_k} \right) \quad (12)$$

By combining eqs. (10) - (12) we obtain

$$\iiint_{V_0} E_{ijkl} \bar{u}_{i,j} u_{k,l} dV = \iint_{A_L} T_i \bar{u}_i dA \quad (13)$$

E_{ijkl} are the moduli of elasticity of the laminate. Because of the laminate symmetry, the following simplification may be made

$$E_{ijkl} \equiv E_{mn} \quad (14)$$

The subscripts i, j, k , and l are related to m and n as follows

$$\begin{aligned} i=j=1 &\rightarrow m=1 & k=l=1 &\rightarrow n=1 \\ i=j=2 &\rightarrow m=2 & k=l=2 &\rightarrow n=2 \\ i \neq j &\rightarrow m=3 & k \neq l &\rightarrow n=3 \end{aligned} \quad (15)$$

The reduced laminate modulus E_{mn} is given by

$$E_{mn} = \sum_{p=1}^N \frac{h^p}{H} \bar{Q}_{ij}^p \quad (16)$$

where h^p is the thickness of the p -th ply. \bar{Q}_{ij}^p is the transformed reduced stiffness matrix for the p -th ply [6] (Appendix A).

SECTION IV

STRESS ANALYSIS-FINITE ELEMENT METHOD

In order to perform the calculations, the problem shown in Figure 1 was simulated by the geometry given in Figure 4. Because of symmetry the stresses were calculated only in one half of the body. Along the symmetry axis, displacement is allowed only in the x_2 direction. Along the lower edge of the plate, displacement is allowed only in the x_1 direction. The intersection of the symmetry axis and the lower edge is considered to be rigidly fixed.

The surface of the hole is subjected to a surface traction T_i . The parameter T_i is related to the applied load. The spatial distribution of T_i depends on the magnitude of the applied load, on the material properties, and on the geometry in a complex manner. It is extremely difficult, if not impossible, to determine the exact distribution of T_i inside the hole. To overcome this difficulty a cosine normal load distribution was assumed. With this approximation T_i becomes

$$T_i = -\frac{4P}{\pi D} n_i \cos \theta \quad (17)$$

The angle θ is in the x_1-x_2 plane and is measured clockwise from the x_2 axis (Figure 1). For isotropic materials the cosine normal load distribution (eq. 17) was found to represent closely the actual load distribution [7]. Calculations performed by previous investigators also showed that for composite materials the stress distribution inside the body is insensitive to the assumed load distribution [1,3,8]. Therefore, eq. (17) should suffice for the purpose of the present analysis which is to determine the overall strength of the joint.

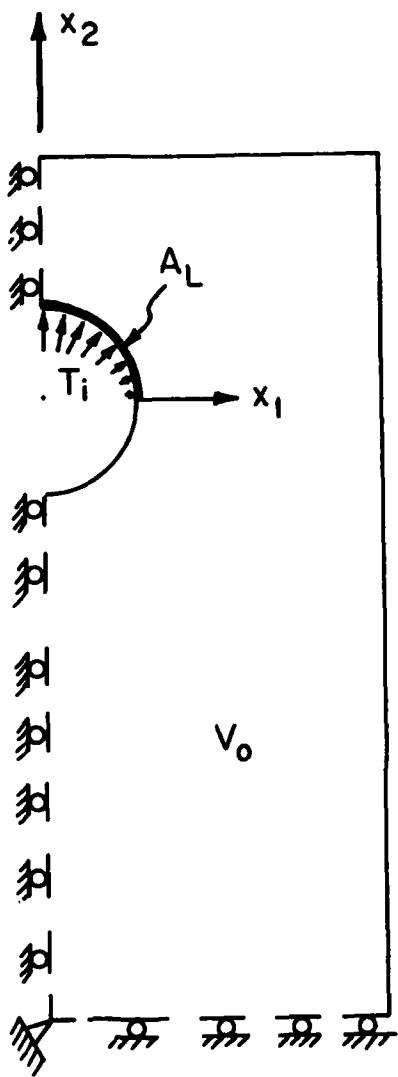


Figure 4. Configuration of a joint approximated in the finite element method

Equations (13) and (17) give

$$\iiint_{V_0} E_{ijkl} \bar{u}_{i,j} u_{k,l} dV = \iint_{A_L} -\frac{4P}{\pi D} n_i \bar{u}_i \cos \theta dA \quad (18)$$

We recall that \bar{u}_i are functions that can be selected arbitrarily. The unknowns in eq. (18) are the displacements \bar{u}_i . Once \bar{u}_i are known the stress at every point can be calculated from eqs (11) and (12). The method of solution of eq. (18) is described below.

Solutions to eq.(18) were obtained by a finite element method (FEM). As a first step in the solution procedure the volume V_0 is subdivided into M subdomains of volume V_g

$$V_0 = \sum_{g=1}^M V_g \quad (19)$$

Equation (18) may now be written as

$$\sum_{g=1}^M \iiint_{V_g} E_{ijkl} \bar{u}_{i,j} u_{k,l} dV = \sum_{g=1}^M \iint_{A_{Lg}} -\frac{4P}{\pi D} n_i \bar{u}_i \cos \theta dA \quad (20)$$

A_{Lg} is the surface of an element on the inside of the hole where the surface traction is applied ($0 \leq \theta \leq \pi/2$). At any surface where load is not applied, A_{Lg} is zero. Advantage is taken now of the assumption that the strains (ϵ_{11} , ϵ_{22} and ϵ_{12}) are independent of the thickness, i.e. the strains are independent of x_3 and depend only on x_1 and x_2 . Thus, the three dimensional grid consisting of M volume elements may be replaced by a two dimensional grid consisting of M surface elements of area s_g (Figure 5)

$$s = \sum_{g=1}^M s_g \quad (21)$$

Equation (20) thus becomes

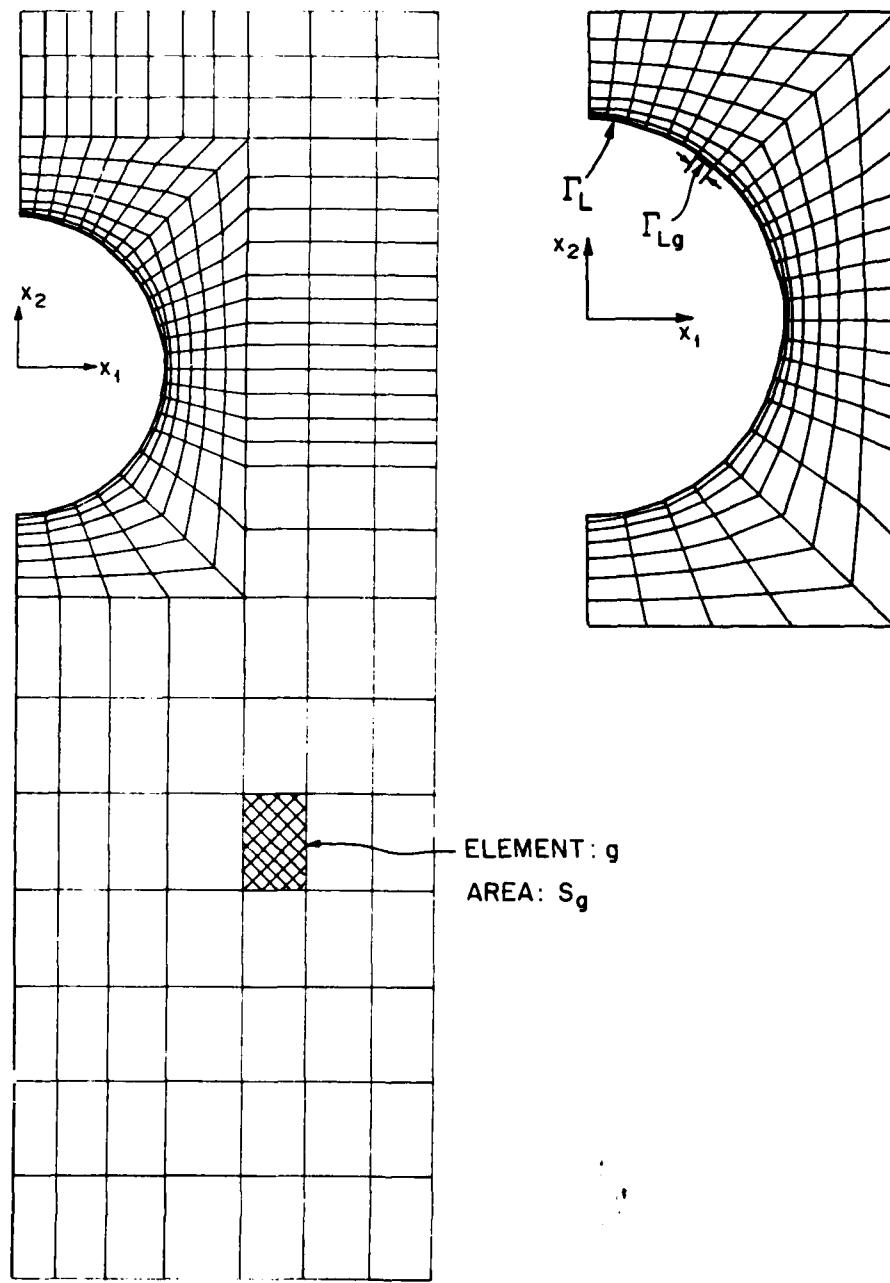


Figure 5. Grid used in the finite element method. Right hand figure is an enlarged view of the grid around the hole

$$\sum_{g=1}^M \iint_{S_g} E_{ijkl} \bar{u}_{i,j} u_{k,l} ds = \sum_{g=1}^M \int_{\Gamma_{Lg}} -\frac{4P}{\pi D} n_i \bar{u}_i \cos\theta d\Gamma \quad (22)$$

Γ_L is a line along the boundary of the hole where the surface traction is applied ($0 \leq \theta \leq \pi/2$, Figure 5). Each segment of this line (denoted by Γ_{Lg}) coincides with the boundary of an element g adjacent to the hole. For those elements which do not lie along this line Γ_{Lg} equals zero. Isoparametric 4-node elements were used in this investigation. The grid was generated using a grid generator [9]. The grid sizes were unequal. Smaller grids were used in the vicinity of the hole to obtain a better resolution of the stresses. Utilizing the symmetry about the x_2 axis, a grid (consisting of 306 elements) was placed on one half of the laminate, as illustrated in Figure 5.

The displacement in each element can be expressed in terms of the displacements of the four nodal points [10]

$$u_i = N_\alpha q_{i\alpha} \quad (23)$$

$$\bar{u}_i = N_\alpha \bar{q}_{i\alpha}$$

The subscript α designates the nodal points ($\alpha = 1, 2, 3$, or 4). N_α is the shape function described in detail in Appendix B. $q_{i\alpha}$ is the displacement at the nodal point α in the i direction.

We define a stiffness matrix for the g -th element as

$$K_{i\beta k\alpha}^g \equiv \iint_{S_g} E_{ijkl} N_{\alpha,l} N_{k,j} ds \quad (24)$$

$K_{i\beta k\alpha}^g$ is an eight by eight matrix. The subscript β may take on the values 1, 2, 3, and 4. The nodal displacements $q_{k\alpha}$ and $\bar{q}_{i\beta}$ are independent of the surface and line integrations. Accordingly, eqs. (22) - (24) yield

$$\sum_{g=1}^M K_{i\beta k\alpha}^g q_{k\alpha} \bar{q}_{i\beta} = \sum_{g=1}^M \bar{q}_{i\beta} \int_{\Gamma_{Lg}} -\frac{4P}{\pi D} n_i N_\beta \cos\theta d\Gamma \quad (25)$$

The nodal displacements $\bar{q}_{i\beta}$ are arbitrary functions and hence eq. (25) can be written

$$\bar{K}_{i\beta k\alpha} q_{k\alpha} = \bar{F}_{i\beta} \quad (26)$$

where the global stiffness matrix, $\bar{K}_{i\beta k\alpha}$ and the load vector $\bar{F}_{i\beta}$ are given by

$$\bar{K}_{i\beta k\alpha} \equiv \sum_{g=1}^M K_{i\beta k\alpha}^g \quad (27)$$

$$\bar{F}_{i\beta} \equiv \sum_{g=1}^M \int_{\Gamma_{Lg}} -\frac{4P}{\pi D} n_i N_\beta \cos\theta d\Gamma \quad (28)$$

The elements of $\bar{K}_{i\beta k\alpha}$ and the components of the vector $\bar{F}_{i\beta}$ are known. Hence, $q_{k\alpha}$ can be obtained from eq. (26) using the Gaussian elimination method [12]. Once $q_{k\alpha}$ are known the displacements u_i are calculated from eq. (23). A computer code was developed for performing the calculations and for generating solutions (section VI).

SECTION V

PREDICTION OF FAILURE

In order to determine the load at which a joint fails and the mode of failure, the conditions for failure must be established. In this investigation the joint is taken to have failed when the combined stresses have exceeded a prescribed limit in any of the plies along an approximately chosen curve (denoted as the characteristic curve). The combined stress limit is evaluated using the failure criterion proposed by Yamada [4]. The coordinates of the characteristic curve are established by extending Whitney and Nuismer's failure hypothesis [13] (developed for open, unloaded holes) to loaded holes.

1) Failure Criterion

Numerous criteria for failure have been proposed in the past [14,15,16,17]. Although the concepts underlying the different failure criteria may be different, the results of the various criteria are generally quite similar. In this investigation Yamada's failure criterion was adopted [4]. This criterion is based on the assumption that just prior to failure of the laminate every ply has failed due to cracks along the fibers. The validity of this assumption is supported by tests performed during this investigation with 64 ply graphite-epoxy (AS/3501-6) laminates. Yamada's criterion states that failure occurs when the following condition is met in any one of the plies

$$\left(\frac{\sigma_x}{x}\right)^2 + \left(\frac{\sigma_{xy}}{S_c}\right)^2 = e^2 \begin{cases} e < 1 & \text{no failure} \\ e \geq 1 & \text{failure} \end{cases} \quad (29)$$

σ_x and σ_{xy} are the longitudinal and shear stresses in a ply, respectively (x and y being the coordinates parallel and normal to the fibers in the ply). X is the longitudinal tensile strength of the ply. S_c is the shear strength of a symmetric, cross ply laminate which has the same number of plies as the laminate under consideration. As indicated in eq. (29) failure occurs when e is equal to or greater than unity.

2) Failure Hypothesis-Characteristic Curve

The hypothesis is proposed here that failure occurs when in any one of the plies the combined stresses satisfy an appropriately chosen failure criterion at any point on a characteristic curve. The characteristic curve (Figure 6) is specified by the expression

$$r_c(\theta) = D/2 + R_{ot} + (R_{oc} - R_{ot}) \cos \theta \quad (30)$$

The angle θ , measured clockwise from the x_2 axis, may range in value from $-\pi/2$ to $\pi/2$. R_{ot} and R_{oc} are the characteristic lengths for tension and compression [13,18]. These parameters can be determined experimentally by measuring the tensile and compressive strengths of notched laminates. R_{ot} and R_{oc} depend only on the material. Therefore, the coordinates of the characteristic curve also depend only on the material, and are independent of the geometry and the stress distribution.

In this investigation the characteristic curve is used together with the Yamada failure criterion. Accordingly (see eq. 29), failure occurs when the parameter e is equal to or is greater than unity at any point on the characteristic curve

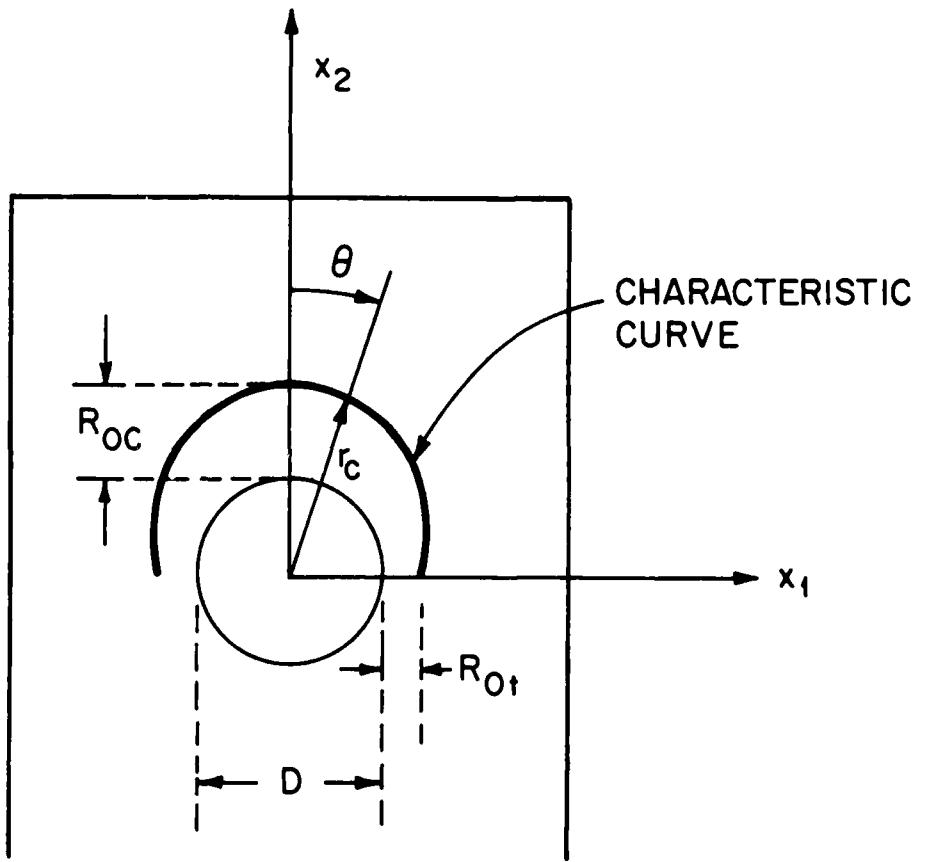


Figure 6. Description of the characteristic curve

$$\left. \begin{array}{ll} \text{No failure} & e < 1 \\ \text{Failure} & e \geq 1 \end{array} \right\} \text{at } r = r_c \quad (31)$$

It is emphasized that the above failure hypothesis is used here in conjunction with the Yamada failure criterion (eq. 29). However, the hypothesis is general, and is not restricted to Yamada's criterion. The characteristic curve proposed here may be used with any other failure criterion.

3) Solution Procedure

Whether or not a joint fails under a given condition is determined as follows. For a given load

a) the stresses ($\sigma_1, \sigma_2, \sigma_{12}$) are calculated in each ply using eqs. (11), (14) and (16), and the FEM described in Sections II, and III,

b) the longitudinal and shear stresses (σ_x, σ_{xy}) are evaluated in each ply employing the transformation

$$\begin{aligned} \sigma_x &= \sigma_1 \cos^2 \eta + \sigma_2 \sin^2 \eta + 2\sigma_{12} \sin \eta \cos \eta \\ \sigma_{xy} &= -\sigma_1 \sin \eta \cos \eta + \sigma_2 \sin \eta \cos \eta + \sigma_{12} (\cos^2 \eta - \sin^2 \eta) \end{aligned} \quad (32)$$

where η is the angle measured counter clockwise from the x_1 -axis to the x -axis of each ply.

c) the parameter e is calculated (eq. 29) along the characteristic curve

d) if e equals or exceeds the value of unity ($e \geq 1$) in any ply along the characteristic curve, the joint is taken to have failed.

The procedure outlined above is used to predict whether or not failure occurs under a given load. Due to the assumption of a cosine normal load distribution around the hole (eq. 17), the calculated

stresses are linearly proportional to the applied load P . This fact together with Yamada's failure criterion (eq. 29) gives

$$P \sim e$$

This relationship is utilized to determine the maximum load (P_{\max}) which can be imposed on the joint. For a given load P , values of e are calculated on the characteristic curve as discussed above (points a-d). The highest value of e (e_o) is then determined, and the maximum load is calculated by the expression

$$P_{\max} = \frac{P}{e_o} \quad (34)$$

The calculation procedure described in the foregoing also provides the location (angle θ_f) at which e first reaches the value of unity ($e=1$) on the characteristic curve. (Figure 7). A knowledge of θ_f provides an estimate of the mode of failure. When θ_f is small ($\theta_f \approx 0^\circ$) failure is by the bearing mode. When $\theta_f \approx 45^\circ$ failure is due to shearout. When $\theta_f \approx 90^\circ$ failure is caused by tension.

In summary

$$\begin{aligned} -15^\circ < \theta_f < 15^\circ & \text{ bearing mode} \\ 30^\circ < \theta_f < 60^\circ & \text{ shearout mode} \\ 75^\circ < \theta_f < 90^\circ & \text{ tension mode} \end{aligned} \quad (35)$$

At intermediate values of θ_f failure may be caused by a combination of these modes.

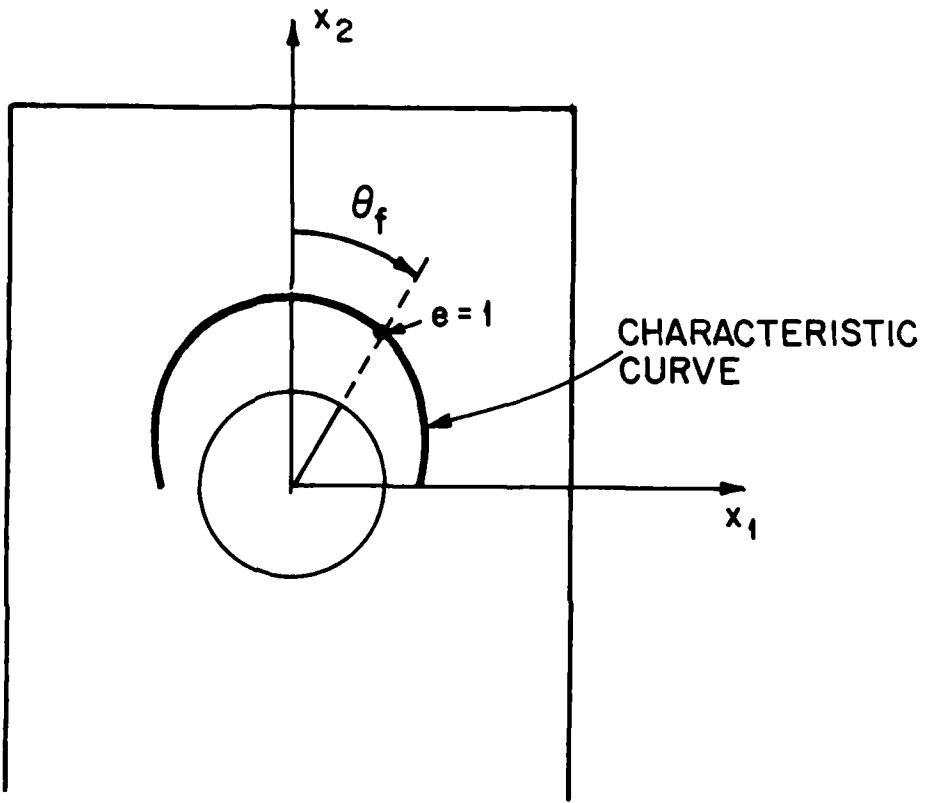


Figure 7. Location of failure ($e=1$) along the characteristic curve

SECTION VI

NUMERICAL SOLUTION

A computer code (designated as BOLT) was developed which is suitable for generating solutions to the problem formulated in Sections II-V. The required input parameters and the output provided by the code are summarized in Table 1.

A Fortran listing of the code and a sample input-output are included in Appendix C.

Table 1. Input parameters required by the computer code and the output provided by the code.

Input Parameters

- 1) Ply properties
 - a) Young's modulus, E_1
 - b) Shear modulus, G_{12}
 - c) Poisson's ratio, ν_{12}
- 2) Ply orientations
- 3) Geometry
 - a) hole diameter, D
 - b) thickness, H
 - c) width, W
 - d) length, L
 - e) edge distances, E
- 4) Characteristic lengths, R_{ot} and R_{oc}
- 5) Longitudinal tensile strength of each ply, X
- 6) Shear strength of cross ply laminate S_c

Output Parameters

- 1) Failure load
- 2) The failure mode
- 3) Stresses in the laminate

SECTION VII

RESULTS AND DISCUSSIONS

Results were generated in order to assess the validity and accuracy of the method and the computer code and to compare the results of the present method with the results of other existing methods of solutions. In addition, parametric studies were performed to evaluate the major characteristics of bolted joints.

1) Isotropic and Orthotropic Plates

Stress distributions were calculated in isotropic plates containing both unloaded (open) and loaded holes and in orthotropic plates containing unloaded holes. These problems were selected because analytical solutions are available for comparisons with the results of the present method.

An analytical solution for the stress distribution in an infinite ($W \rightarrow \infty$) isotropic plate containing an unloaded hole was given by Timoshenko [19]. The stress distribution in such a plate was also calculated by the present method. The parameters used in the numerical calculations are given in Figure 9. A large width ($W/D=14$) was used in the calculation to approximate an infinite plate. The results of the present method and the analytical solution of Timoshenko are compared in Figure 8. There is excellent agreement between the stresses calculated by the two methods.

The stresses in isotropic plates containing loaded holes were also calculated. Plates of infinite and finite widths were considered. Calculations were performed for the parameters given in Figure 9 and Table 2. From the calculated stresses, the stress concentration factor was determined. The stress concentration factor is defined as

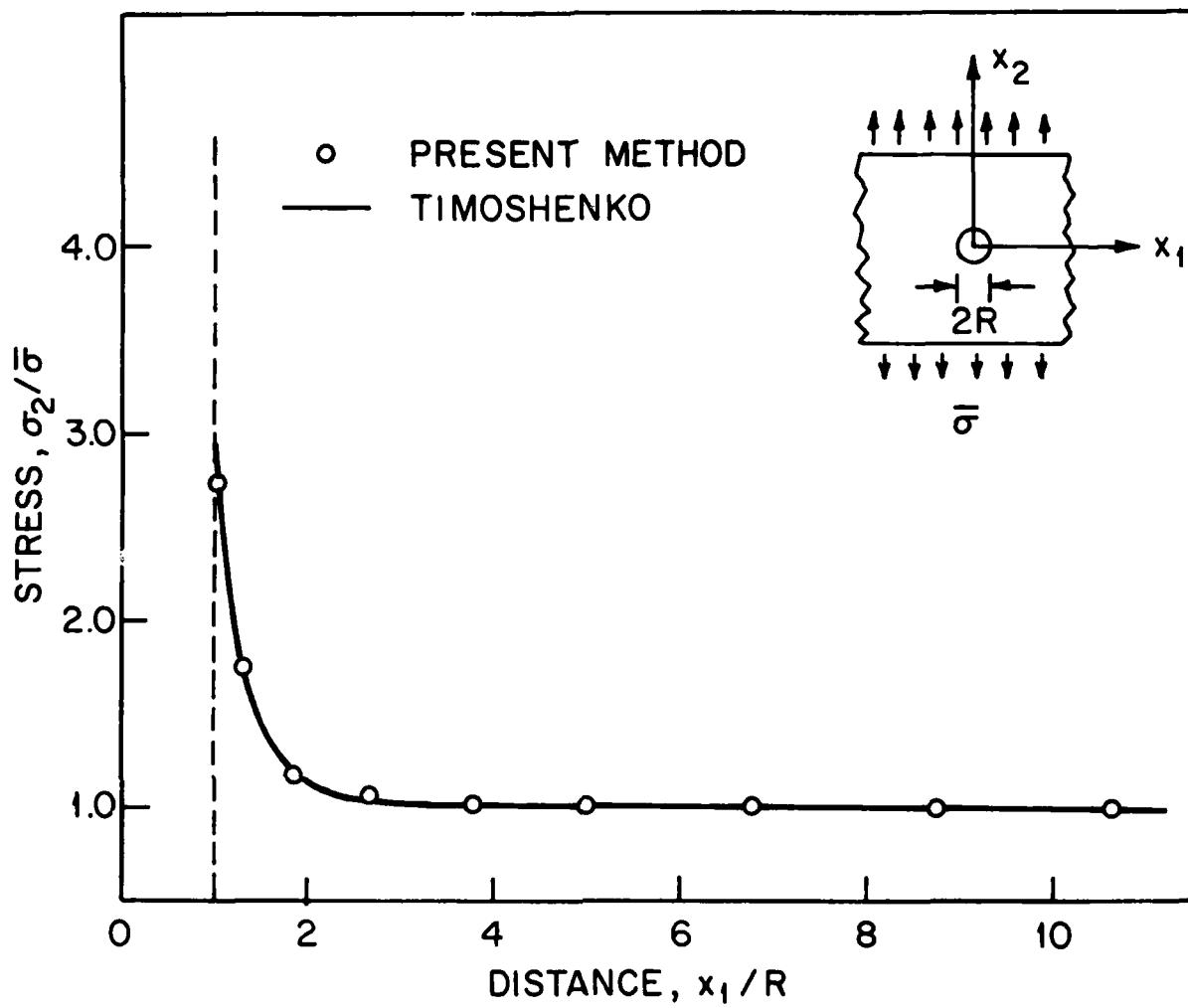


Figure 8. The stress σ_2 along the x_1 -axis in an isotropic infinite plate containing a circular hole. Comparison of the present results with the theoretical results given by Timoshenko [19]. Parameters used in the numerical calculations: $\bar{\sigma}=1.64$ MPa, $D=2R=7.62$ mm, $W/D=14$, $E/D=14$, $L/D=28$

$$\text{SCF} \equiv \frac{(\sigma_2)_{\text{max}}}{B} \quad (36)$$

where $(\sigma_2)_{\text{max}}$ is the maximum stress in the plate (perpendicular to the x_1-x_3 plane, Figure 1) and B is the bearing stress

$$B \equiv \frac{P}{(H)(D)} \quad (37)$$

The stress concentration factors obtained by the present method were compared to those reported by previous investigators (Table 2). The maximum difference in the stress concentration factors given by the different methods is about 20 percent. The stress concentration factor given by the present method differs from the values given by previous investigators at most by 15 percent.

The stresses in an isotropic plate of finite width containing a loaded hole are shown in Figure 9. The stresses calculated by the present method are in excellent agreement with De Jong's approximate solution [8].

The stress distribution in an orthotropic plate of finite width containing an open (unloaded) hole was also calculated. The calculations were performed for a plate with the symmetric laminate lay up of $[0/90]_S$. An analytical solution for this problem was provided previously by Nuismer and Whitney [18], who modified Lekhnitskii's earlier solution [20] for an infinite plate. The results given in Figure 10 show excellent agreement between the stresses calculated by the present method and by the analytical solution.

The aforementioned comparisons indicate that the present method predicts the stress distribution around loaded and unloaded holes with high accuracy.

Table 2. Stress Concentration Factor (SCF) Around a Pin Loaded Hole
Contained in an Isotropic Plate of Infinite Width.
Comparison of Present Result with Those Obtained by
Previous Investigators.

<u>Investigators</u>	<u>SCF</u>
Present Result*	0.985
Hong [22]	0.955
De Jong [8]	1.058
Eshwar et al [23]	0.922
Bickley [7]	0.81

*Present results was calculated for the case:

$D = 7.62 \text{ mm}$, $W/D = 8$, $E/D = 4$, $L/D = 14$.

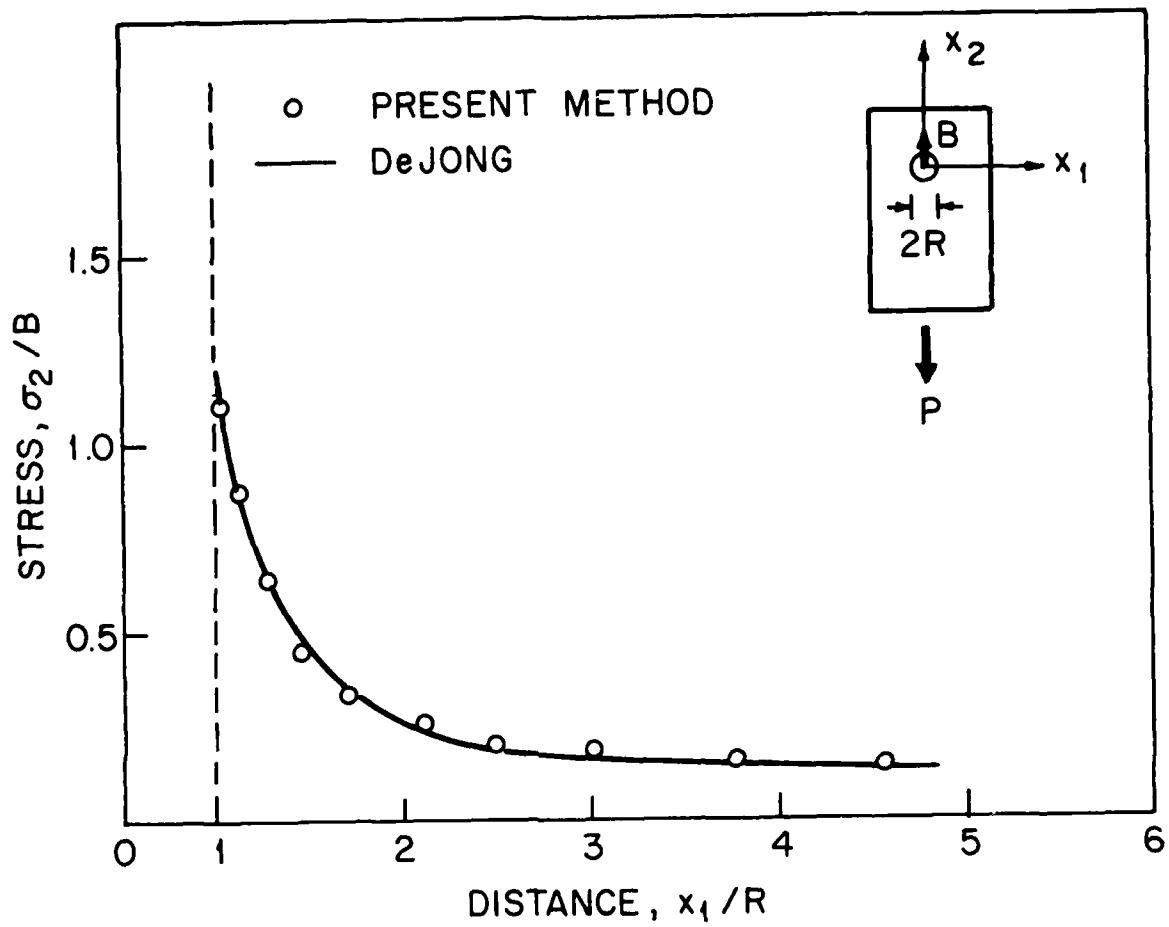


Figure 9. The stress σ_2 along the x_1 -axis in an isotropic plate of finite width containing a loaded hole. Comparison of the present results with the theoretical results given by De Jong [8]. Parameters used in the numerical calculations: $D=7.62$ mm, $W/D=5.0$, $E/D=4.0$, $L/D=14.0$

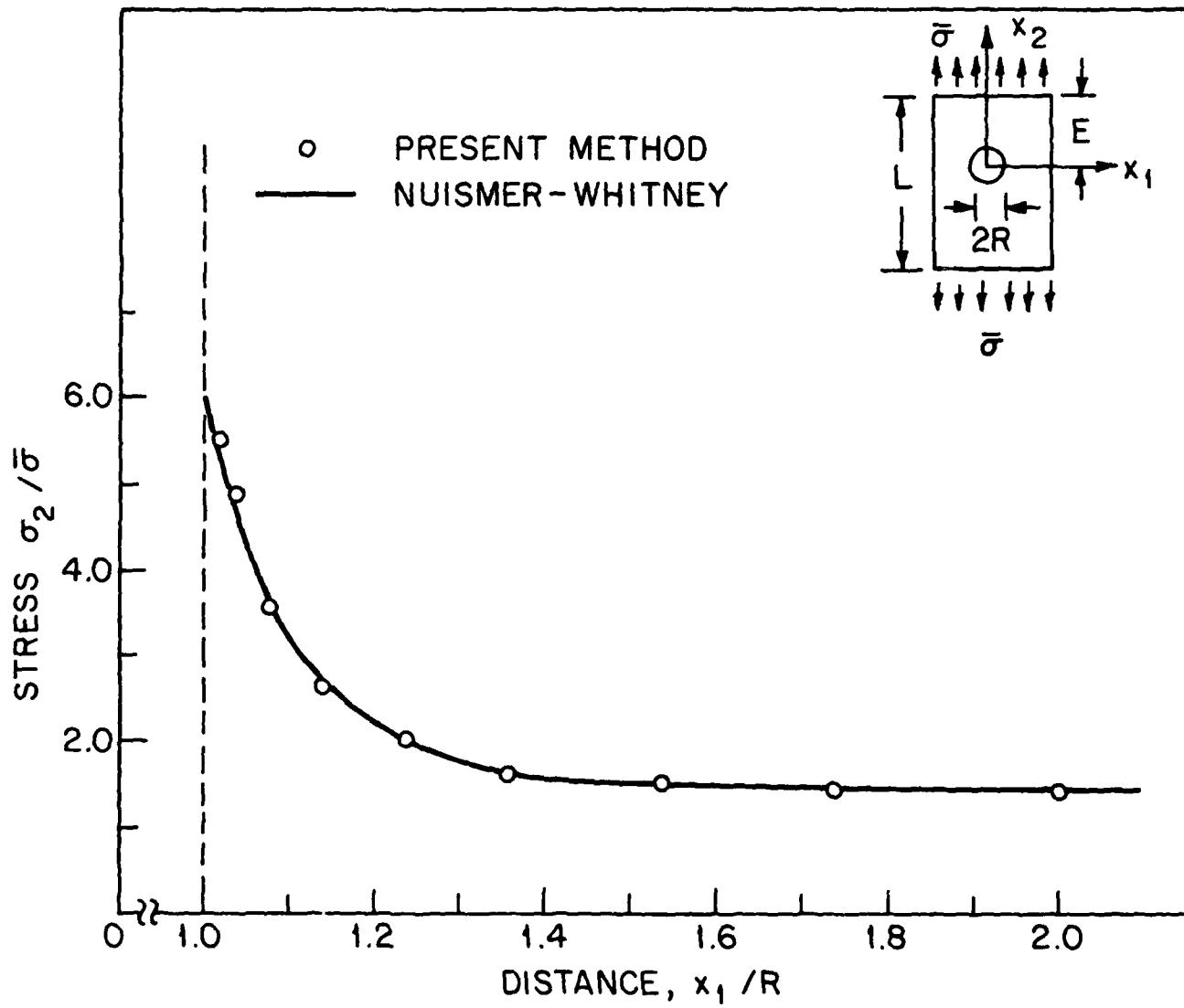


Figure 10. The stress σ_2 along the x_1 -axis in an orthotropic finite plate $[0/90]_s$ containing a circular hole. Comparison of the present results with the theoretical results obtained by Nuismer and Whitney [18]. Parameters used in the numerical calculations: Material: Graphite/Epoxy T300/5208, $E_1=149.8$ GPa, $E_2=11.2$ GPa, $G_{12}=5.39$ GPa, $\nu_{12}=0.29$, $\bar{\sigma}=2.3$ MPa, $D=24.5$ mm, $W/D=3.0$, $E/D=4.0$, $L/D=14.0$.

2) Failure Strength and Failure Mode

Failure strengths of mechanically fastened composite joints calculated by the present method were compared to available data and to failure strengths predicted by other numerical methods.

Failure strengths of joints in graphite-epoxy laminates were measured by Van Siclen [21] and by Garbo and Oganowski [3]. The test conditions are summarized in Table 3. The failure strengths of the joints under these conditions were also calculated by the present method. The material properties used in the calculations are summarized in Table 4. Comparisons between the experimental and predicted failure strengths are given in Table 5. In these tables comparisons between the data and the failure strengths predicted by Agarwal [2] and by Garbo and Oganowski [3] are also included.

As can be seen, of the three analytical methods included in Table 5 the present one predicts the failure most accurately. In most cases, the failure load given by the present method agrees with the data within about 10%. On the other hand, the failure loads given by the Agarwal and by the Garbo and Oganowski methods are in error by as much as 50 percent.

One point is of interest here. When there is a high fraction of zero degree plies in the laminate (parallel to the body direction) the laminate fails in the shearout mode. In this case the calculated results are sensitive to the value of the laminate shear strength S_c . For example, when the S_c value obtained with cross ply laminates is used to calculate the failure load of $[0/\pm 45/90]_S$ laminates containing 70 percent of 0° plies the calculated and

Table 3. Summary of Test Conditions

<u>Investigator</u>	<u>Material</u>	<u>Case No.</u>	<u>D (mm)</u>	<u>W/D</u>	<u>E/D</u>	<u>L/D</u>	<u>H (mm)</u>	<u>Lay up</u>	<u>Volume Fraction</u>
									<u>80°</u> <u>845°</u> <u>890°</u>
Van Siclen [21]	T300/sp286	1	4.76	8.025	2.983	14.68	1.067	[0/±45/90] _S	25 50 25
		2	4.76	8.025	2.083	14.68	1.067	[0/±45/90] _S	25 50 25
		3	4.76	5.336	2.083	14.68	2.235	[0/±45/90] _S	25 50 25
		4	4.76	5.336	2.983	14.68	1.397	[0 ₂ /±45/90] _S	40 40 20
		5	4.76	5.336	2.983	14.68	1.067	[0/±45/90] _S	25 50 25
		6	4.76	8.025	4.013	14.68	1.067	[0/±45/90] _S	25 50 25
		7	4.76	5.336	2.983	14.68	1.118	[±45] _{2S}	0 100 0
		8	4.76	5.336	2.983	14.68	1.118	[0/90] _{2S}	50 0 50
		9	4.76	5.336	2.983	14.68	1.067	[0 ₂ /±45] _S	50 50 0
Garbo and Ogonowski [3]	AS/3501-6	10	6.35	6	3	14.68	5.283	[0/±45/90] _S	30 60 10
		11	6.35	6	3	14.68	5.283	[0/±45/90] _S	50 40 10
		12	6.35	6	3	14.68	5.283	[0/±45/90] _S	70 20 10

Table 4. Material Properties Used in the Calculations

<u>Ply Properties</u>	<u>T300/SP286</u>	<u>AS/3501-6</u>
Longitudinal Modulus E_1 Gpa(10 ksi)	130 (18.7)[21]	130(18.85)[3]
Transverse Modulus E_2 Gpa(10 ksi)	8.274(1.2)[21]	13.1(1.9)[3]
Shear Modulus G_{12} Gpa(10 ksi)	5.033(0.73)[21]	5.86(0.85)[3]
Poisson Ratio ν_{12}	0.30[21]	0.30[3]
Tensile Strength X Gpa(10 ksi)	1.23(0.178)[21]	1.58(0.23)[3]
Shear Strength S Gpa(10 ksi)	0.05(0.0073)[21]	0.12(0.017)[3]
<u>Laminate Properties</u>	<u>T300/SP286</u>	<u>AS/3501-6</u>
Cross-ply Laminate Shear Strength S_c Gpa(10 ksi)	0.125 (a)(0.018)	0.204 (b)(0.03) 0.12(c)(0.017)
Characteristic Length R_{ot} mm(in)	1.092 (d)(0.043)	0.584 (0.023)[3]
Characteristic Length R_{oc} mm(in)	3.048 (e)(0.12)	1.727(e)(0.068)

- (a) Tests with Glass/Epoxy showed S_c to be about 2.5 times higher than the ply shear strength S [4]. The 2.5 multiplier was used to obtain S_c of Graphite/Epoxy T300/SP286 from the ply shear strength S given in [21].
- (b) The value of S_c for AS/3501-6 was taken to be 1.7 times the ply shear strength S .
- (c) This value was used for laminates containing more than 70% (by volume) of 0 degree plies.
- (d) For T300/SP286 the value was chosen from [18] for T300/5208.
- (e) The values of R_{oc} for T300/SP286 and AS/3501-6 were evaluated by the method given in [18] together with the data in [24,25].

Table 5. Comparisons Between the Experimental (P) and Calculated (P_C) Failure Loads. Case Numbers Correspond to Test Conditions Given in Table 3.

<u>Material</u>	<u>Case No.</u>	<u>Lay Up</u>	<u>Percent Difference</u> (1-P/P _C)x100	
			<u>Present Results</u>	<u>Agarwal(1980)</u>
T300/SP286	1	[0/±45/90] _S	3.7	8.7
	2	[0/±45/90] _S	0.01	0.01
	3	[0/±45/90] _{2S}	1.81	7.78
	4	[0 ₂ /±45/90] _S	0.01	0.01
	5	[0/±45/90] _S	8.66	1.16
	6	[0/±45/90] _S	11.18	1.2
	7	[±45] _{2S}	11.1	49.43
	8	[0/90] _{2S}	12.3	49.85
	9	[0 ₂ /±45] _S	7.64	20.0
AS/3501-6	10	[0/±45/90] _S	0.5	45
	11	[0/±45/90] _S	3.8	27
	12	[0/±45/90] _S	15.0	14

Present Results Garbo and Ogonowski(1981)

measured values differ by 35 percent. On the other hand, if the shear stress of the individual ply is used in the calculations the difference between the calculated failure load and the data is only 15 percent. This indicates that the value of S_c must be selected carefully.

Failure loads calculated by Waszczak and Cruse [1] are compared to data in Table 6. Waszczak and Cruse's method also yields failure loads which are in error by as much as 50 percent. The present method was not applied to Waszczak and Cruse's data because the material properties needed for the calculation were unavailable.

It is interesting to note that the accuracies of all four methods (present, Agarwal, Garbo and Ogonowski, and Waszczak and Cruse) depend on the arrangements of the plies in the laminate. In general, the analytical predictions are most accurate for quasi-isotropic laminates, and are least accurate for angle ply and cross ply laminates. However, even for angle ply and cross ply laminates the present method yields results within about 10 percent accuracy, in contrast with the results of other existing methods of solutions, which may be in error by as much as 50 percent.

The failure modes predicted by the present method were also compared to failure modes observed experimentally. These comparisons, given in Table 7, show that the present method predicts well the mode of failure.

The aforementioned comparisons between the results of the present method and the data show that the method predicts with good accuracy both the load at which the joint fails and the mode of failure. The good agreements between the predictions of the model and the data

Table 6. Comparisons Between the Experimental Failure Loads and the Values Predicted by Waszczak and Cruse [1]

Material	Lay Up	Volume Fraction %			E/D	Percent Difference (1-[P/Pc])x100
		45	0	90		
Graphite/Epoxy	[±45]	100	0	0	4	53
Boron/Epoxy	[(±45/0)S/90]S	72.3	18.2	9.1	4	24
	[±45/90]S	62.5	0	37.5	4	3
	[±45/(0/90)S]S	13.3	90.0	6.7	6	42
	[06/±455]	62.5	37.5	0	7.5	2.5
	[06/±45]	62.5	37.5	0	7.35	4

Table 7. Comparisons of Predicted Failure Modes with those Observed Experimentally
 T-Tension Mode S-Shearout Mode B-Bearing Mode

Material	Case No.	Lay Up	Observed Failure Mode		Predicted Present	Failure Mode Agerwal(1980)
			Present	Agerwal(1980)		
T300/SP286	1	$[0/\pm 45/90]_S$	S/R	S	S	S
	2	$[0/\pm 45/90]_S$	S	S	S	S
	3	$[0/\pm 45/90]_2S$	T/S	T	T	T
	4	$[0_2/\pm 45/90]_S$	S	S	S	S
	5	$[0/\pm 45/90]_S$	T	T	T	T
	6	$[0/\pm 45/90]_S$	B	B	B	B
	7	$[0/90]_2S$	S	S	S	S
	8	$[0_2/\pm 45]_S$	B/S	S	S	S
	9					
AS/3501-6					Present	Garbo & Ogonowski (1981)
	10	$[0/\pm 45/90]_S$	B/S	B		-
	11	$[0/\pm 45/90]_S$	B/S	S		-
	12	$[0/\pm 45/90]_S$	B/S	S		-

(illustrated in Tables 5-7) create further confidence in the model.

3) Effects of Geometry and Ply Orientations

Parametric studies were performed to evaluate the effects of joint geometry and ply orientation on the failure strength and on the failure mode.

The effects of joint width on joint failure is illustrated in Figure 11. In this, and in subsequent figures, the failure load is normalized with respect to the ultimate tensile load of the laminate (without the hole) in the direction of the applied load. As is shown by the results in Figure 11, in general, the maximum load the joint can carry decreases as the hole size decreases, when the width to hole diameter ratio is greater than about 3. As the hole diameter approaches the width ($W/D \rightarrow 1$) the strength reduces to zero ($P \rightarrow 0$). Here failure loads were not calculated for W/D less than three because at such low W/D ratios the assumption of the cosine load distribution (eq. 17) is inaccurate [8].

The effect of edge distance E (Figure 1) on the failure load is shown in Figure 12. For the lay-ups $[0/\pm 45/90]_s$ and $[0/90]_{2s}$ increasing edge distance results in higher failure loads, as long as E/D is less than about 4. For higher edge ratios ($E/D > 4$) an increase in edge distance does not seem to influence significantly the failure load. For the lay-up $[0_2/\pm 45]_s$, the failure load does not vary significantly with E/D .

The effects of ply orientation on the failure load are given in Figure 13. This figure illustrates the effects of two parameters 1) the maximum ply angle ϕ in the laminate and 2) the change in

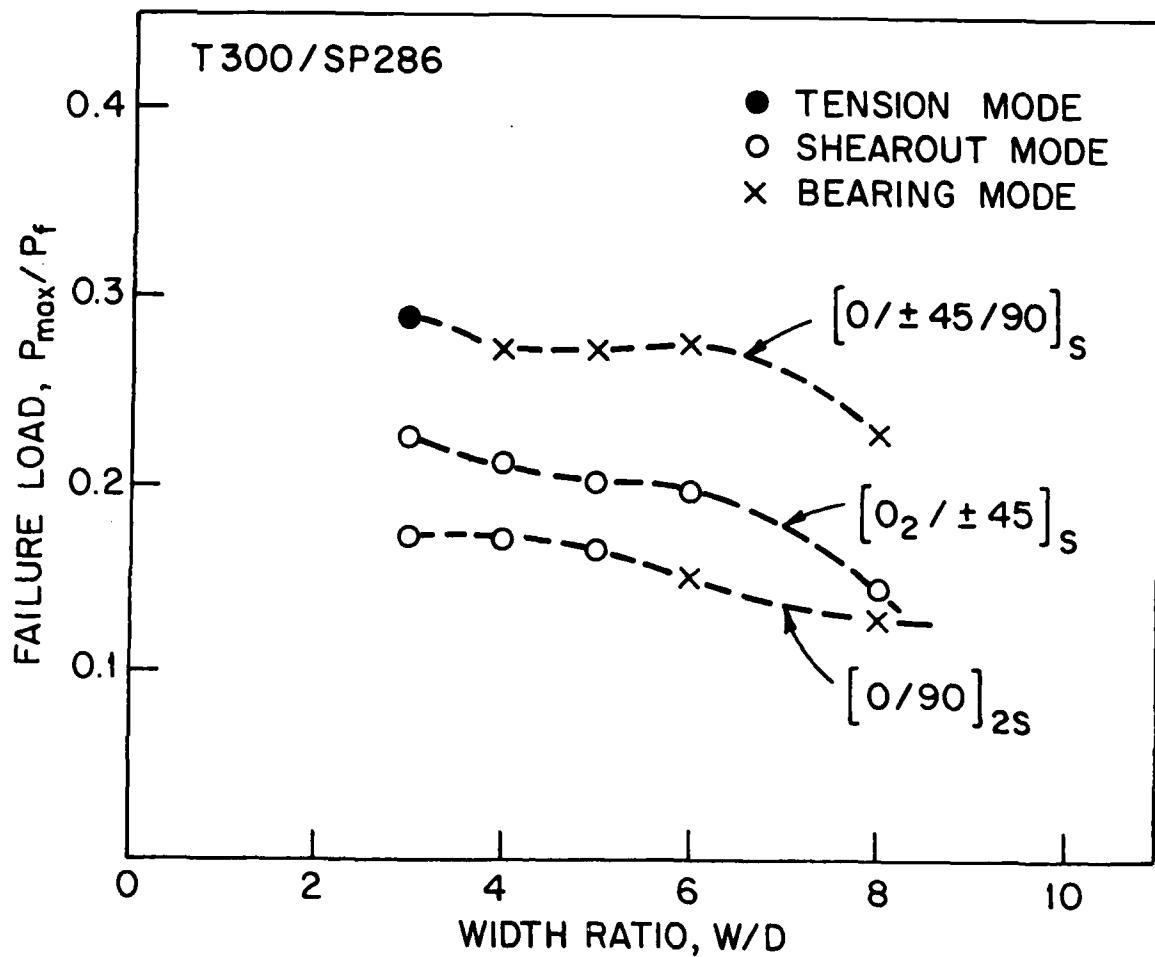


Figure 11. The effects of width ratio on the failure load of laminates with different ply orientations. P_f is the tensile failure load of laminates without holes. Parameters used in the numerical calculations:
 Material: Graphite/Epoxy T300/SP286, $W=38$ mm, $E=50.8$ mm,
 $L=203.2$ mm, $H=1.067$ mm for $[0/\pm 45/90]_s$ and
 $[0_2/\pm 45]_s$, and $H=1.18$ mm for $[0/90]_{2s}$

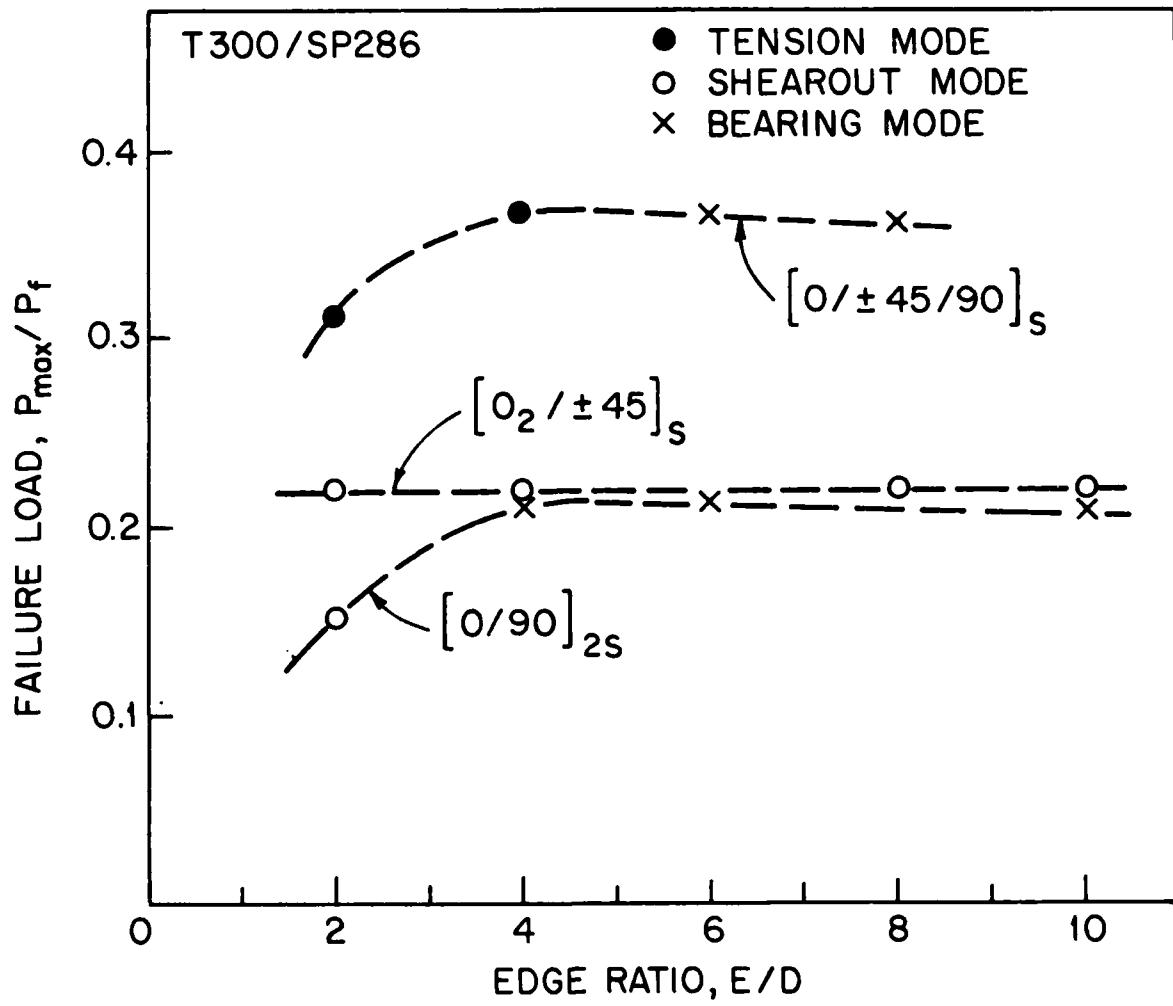


Figure 12. The effects of edge ratio on the failure load of laminates with different ply orientations.
 Parameters used in the numerical calculations:
 Material: Graphite/Epoxy T300/SP286, D=5.08 mm,
 W/D=5, L/D=14

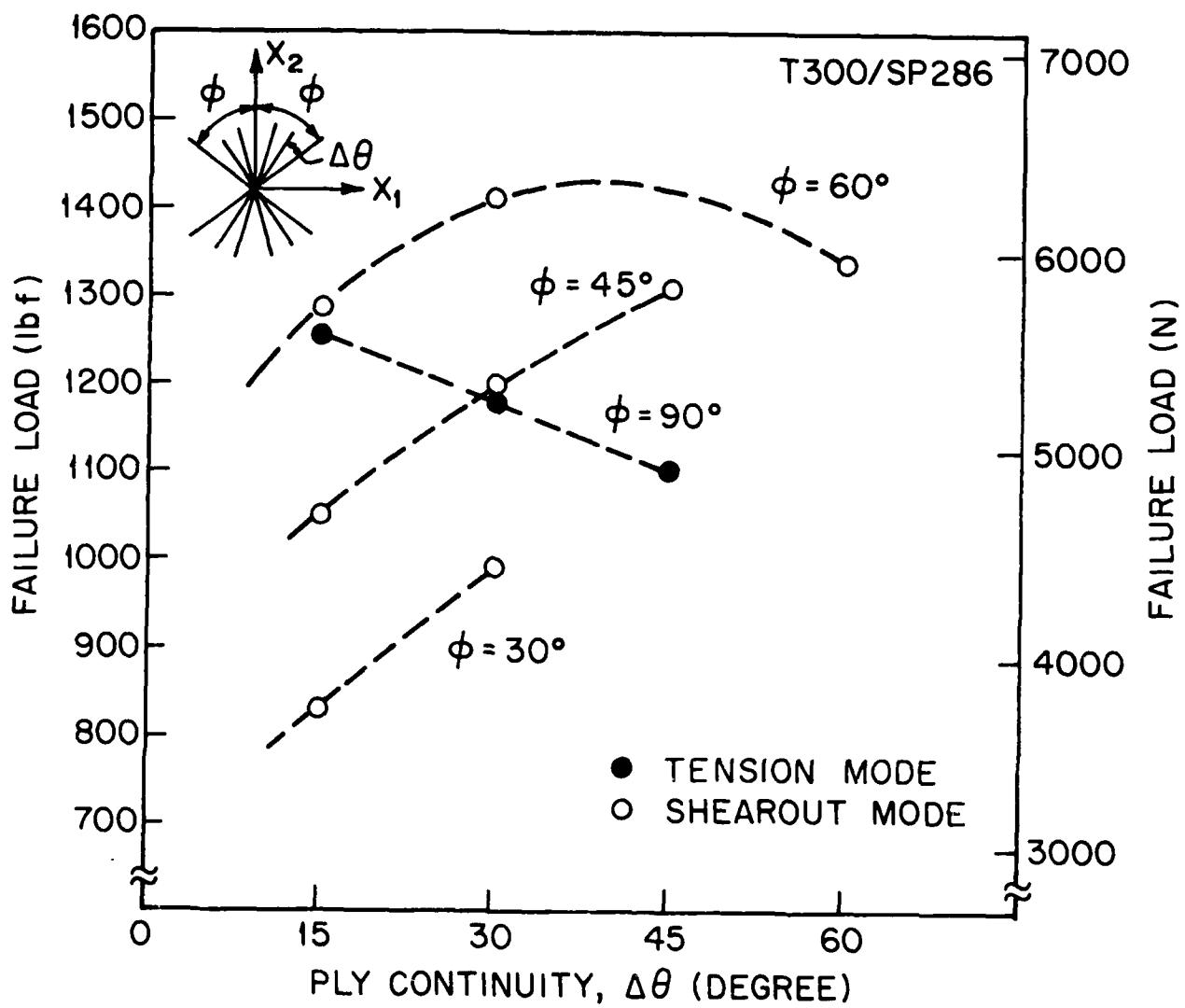


Figure 13. The effects of maximum ply angle ϕ and ply continuity $\Delta\theta$ on the failure load of mechanically fastened joints. Parameters used in the numerical calculations: Material: Graphite/Epoxy T300/SP286, $D=4.76$ mm, $W/D=5.336$, $E/D=2.983$, $L/D=14.68$, $H=1.397$ mm

orientation between two adjacent plies $\Delta\theta$. The latter parameter is referred to here as "ply continuity". The results in Figure 13 show that the failure load increases both with increasing ϕ and with increasing $\Delta\theta$, as long as failure is by shearout mode. On the other hand, the failure load decreases with increasing ϕ and with increasing $\Delta\theta$ when the failure is by tension mode. These results indicate that care must be exercised in designing bolted joints. If there are no other design constraints, the range of ply orientation ϕ and the ply continuity $\Delta\theta$ should be determined with the use of the computer code such that the joint can withstand the highest load.

SECTION VIII

CONCLUDING REMARKS

The model and computer code developed in this investigation can be used in the design of mechanically fastened joints involving fiber reinforced laminates. The computer code can be used to determine

- a) the optimum geometry of a joint for a given load,
- b) whether or not the joint will fail under a given load,
- c) the failure load,
- d) the mode of failure, and
- e) the ply in which failure first occurs.

The good accuracy of the method suggests that it might be worth it to extend the method to joints consisting of two or more fasteners.

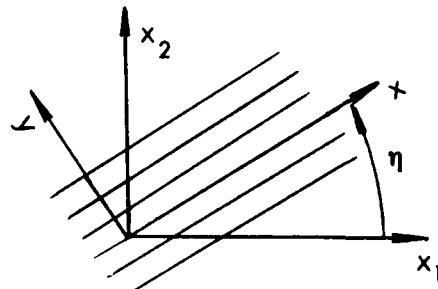
The results of parametric studies performed with the present computer code show that the material properties, joint geometry, and ply orientation, all effect significantly the strength of mechanically fastened joints.

REFERENCES

1. J.P. Waszczak and T.A. Cruse, "Failure Mode and Strength Predictions of Anisotropic Bolt Bearing Specimens", *J. of Composite Materials*, Vol. 5, 1971, pp. 421-425.
2. B.L. Agarwal, "Static Strength Prediction of Bolted Joint in Composite Material", *AIAA Journal*, Vol. 18, 1980, pp. 1371-1375.
3. S.P. Garbo and J.M. Ogonowski, "Effect of Variances and Manufacturing Tolerances on the Design Strength and Life of Mechanically Fastened Composite Joints", *Flight Dynamics Laboratory, Air Force Wright Aeronautical Laboratories, Technical Report AFWAL-TR-81-3041*, April, 1981.
4. S.E. Yamada, "Analysis of Laminate Strength and Its Distribution", *J. of Composite Materials*, Vol. 12, 1978, pp. 275-284.
5. Y.C. Fung, Foundations of Solid Mechanics, Prentice-Hall, Inc. Englewood Cliffs, New Jersey, 1965.
6. R.M. Jones, Mechanics of Composite Materials, Scripta Book Company, Washington, D.C., 1975, p. 51
7. W. Bickley, "The Distribution of Stress Round a Circular Hole in a Plate", *Phil. Trans. Roy. Soc., A(London)*, Vol. 227, 1922 pp. 383-415.
8. T. De Jong, "Stress Around Pin-Loaded Holes in Elastically Orthotropic or Isotropic Plates", *J. of Composite Materials*, Vol. 11, 1977, p. 313-331.
9. N. Kikuchi, "Class Notes for Finite Element Methods", University of Michigan, Fall 1981.
10. O.C. Zienkiewicz, The Finite Element Method, McGraw-Hill Book Co., New York, 1977.
11. C.S. Desai and J.F. Abel, Introduction to the Finite Element Method, Litton Educational Publishing, Inc., 1972.
12. P. Tong and J.N. Rossettos, Finite Element Method-Basic Technique and Implementation, The MIT Press, 1977.
13. J.M. Whitney and R.J. Nuismer, "Stress Fracture Criteria for Laminated Composite Containing Stress Concentrations", *J. of Composite Materials*, Vol. 8, 1974, pp. 253-265.
14. S.W. Tsai, "Strength Theories of Filamentary Structures", in Fundamental Aspects of Fiber Reinforced Plastic Composites R.T. Schwartz and H.S. Schwartz (eds.), Wiley Interscience,

- New York, 1968, pp. 3-11.
15. S.W. Tsai and F.M. Wu, "A General Theory of Strength for Anisotropic Materials", *J. of Composite Materials*, Vol. 5, Jan. 1971, pp. 58-80.
 16. S.W. Tsai, "Mechanics of Composite Materials, Part II-Theoretical Aspects", Air Force Material Laboratory. Technical Report, AFML-TR-66-149, 1966.
 17. O. Hoffman, "The Brittle Strength of Orthotropic Materials", *J. of Composite Materials*, Vol. 1, 1967, pp 200-206.
 18. R.J. Nuismer and J.M. Whitney, "Uniaxial Failure of Composite Laminates Containing Stress Concentrations", *Fracture Mechanics of Composites*, ASTM STP 593, 1975, pp. 117-142.
 19. S. Timoshenko, and S. Woinowsky-Krieger, Theory of Plates and Shells, McGraw-Hill, New York, 1940.
 20. S.G. Lekhnitskii, Anisotropic Plates, (translated from the Second Russian Edition by S.W. Tsai and T. Cheron), Gordon and Breach, Science Publisher, Inc., New York, 1968.
 21. R.C. Van Siclen, "Evaluation of Bolted Joints in Graphite/Epoxy", *Proceedings of the Army Symposium on Solid Mechanics: Role of Mechanics in the Design of Structural Joints*, 1974, pp. 120-138.
 22. C.-S. Hong, "Stresses Around Pin-Loaded Hole in Finite Orthotropic Laminates", *Transactions of the Japan Society for Composite Materials, Trans. JSCE*, Vol. 6, 1980, pp. 50-55.
 23. V.A. Eshwar, B. Dattaguru and A.K. Rao, "Partial Contact and Friction in Pin Joints", ARDB-STR-5010, India, 1977.
 24. R.J. Nuismer and J.D. Labor, "Applications of the Average Stress Failure Criterion: Part I-Tension", *J. of Composite Materials*, Vol. 12, 1978, pp. 238-249.
 25. R.J. Nuismer and J.D. Labor, "Applications of the Average Stress Failure Criterion: Part II-Compression", *J. of Composite Materials*, Vol. 13, 1979, pp. 49-60.

Appendix A - The Transformed Reduced Stiffness Matrix \bar{Q}_{ij}^P



The components of the matrix \bar{Q}_{ij}^P appearing in Eq. (16) are

$$\bar{Q}_{11}^P = Q_{11}^P \cos^4 \eta + 2(Q_{12}^P + 2Q_{66}^P) \sin^2 \eta \cos^2 \eta + Q_{22}^P \sin^4 \eta$$

$$\bar{Q}_{12}^P = (Q_{11}^P + Q_{22}^P - 4Q_{66}^P) \sin^2 \eta \cos^2 \eta + Q_{12}^P (\sin^4 \eta + \cos^4 \eta)$$

$$\bar{Q}_{22}^P = Q_{11}^P \sin^4 \eta + 2(Q_{12}^P + 2Q_{66}^P) \sin^2 \eta \cos^2 \eta + Q_{22}^P \cos^4 \eta$$

$$\bar{Q}_{13}^P = (Q_{11}^P - Q_{12}^P - 2Q_{33}^P) \sin \eta \cos^3 \eta + (Q_{12}^P - Q_{22}^P + 2Q_{33}^P) \sin^3 \eta \cos \eta$$

$$\bar{Q}_{23}^P = (Q_{11}^P - Q_{12}^P - 2Q_{33}^P) \sin^3 \eta \cos \eta + (Q_{12}^P - Q_{22}^P + 2Q_{33}^P) \sin \eta \cos^3 \eta$$

$$\bar{Q}_{33}^P = (Q_{11}^P + Q_{22}^P - 2Q_{12}^P - 2Q_{33}^P) \sin^2 \eta \cos^2 \eta + Q_{33}^P (\sin^4 \eta + \cos^4 \eta)$$

in which

$$Q_{11}^P = \frac{E_1^P}{1 - \nu_{12}^P \nu_{21}^P}$$

$$Q_{12}^P = \frac{\nu_{12}^P E_2^P}{1 - \nu_{12}^P \nu_{21}^P} = \frac{\nu_{21}^P E_1^P}{1 - \nu_{12}^P \nu_{21}^P}$$

$$Q_{22}^P = \frac{E_2^P}{1 - \nu_{12}^P \nu_{21}^P}$$

$$Q_{33}^P = G_{12}^P$$

The superscript p denotes the material properties of the p -th ply and the angle η is measured from the x_1 -axis to the x -axis. E_1^p , E_2^p and G_{12}^p are the longitudinal, transverse and shear moduli of the p -th ply, respectively. ν_{12}^p and ν_{21}^p are Poisson's ratios of p -th ply and satisfy the relation

$$\frac{\nu_{12}^p}{E_1^p} = \frac{\nu_{21}^p}{E_2^p}$$

Appendix B - Shape Function Used in the Finite Element Code.

In the isoparametric element, the geometry and the displacement of the element are described in terms of the shape function N_α by a transformation from a Master Element in the $r-s$ coordinate system to the element in the x_1-x_2 coordinate system (Figure 14)

$$x_i = N_\alpha(r, s) \bar{x}_{i\alpha} \quad i = 1, 2$$

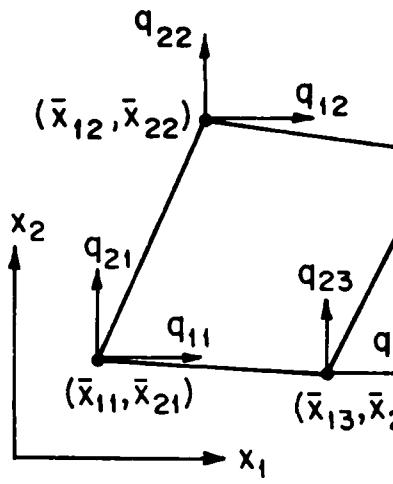
$$u_i = N_\alpha(r, s) q_{i\alpha} \quad \alpha = 1, 2, 3 \text{ or } 4$$

$$N_\alpha(r, s) = 1/4(1+r r_\alpha)(1+s s_\alpha) \quad -1 \leq r, s \leq 1$$

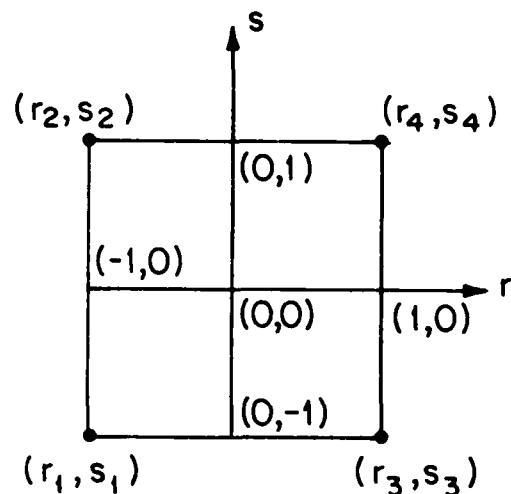
where $x_{i\alpha}$ is the coordinate of node α in the i -direction and $q_{i\alpha}$ is the displacement of node α in the i -direction, and r_α and s_α are the coordinates of node α referred to the Master Element

Note the property

$$N_\alpha(r_\beta, s_\beta) = \begin{cases} 1, & \text{if } \alpha = \beta \\ 0, & \text{if } \alpha \neq \beta \end{cases}$$



Element in x_1 - x_2 coordinates



Master element in local
r-s coordinates

Figure 14. Geometry of an element used in the finite element calculations. Left: Element in the x_1 - x_2 coordinate system. Right: Element (master element) in the local (r-s) coordinate system. x_{ia} is the coordinate of node α in the i direction, q_{ia} is the displacement of node α in the i direction and (r_α, s_α) are the coordinates of node α in the r-s coordinate system, $i=1, 2$, $\alpha=1, 2, 3$ or 4

**Appendix C - Listing of the Computer Code "BOLT", and a Sample
of Input and Output.**

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8)

MAIN 06-01-82 16:14:16 PAGE 001

51

```

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8)          MAIN          06-01-82          16:14:16          PAGE P002

C DO 15 I=1,353
C WRITE 6,400  I, (DIS(J,I),J=1,2)
C 400 FORMAT( /,5X, 'POINT= ',I3,5X, 'X-DIS',E15.8,5X, 'Y-DIS',E15.8,/)
C 15 CONTINUE
C      CALL STRESS(DIS)
C      CALL CFAIL(KT,MT,FAL)
C      IF(FAL.GT. 1.) GO TO 100
C
C      TN=DSORT(1.0/FAL)
C      PF=TN*PF*HI*4.0
C
C      50 CALL OUTPUT(KT,MT,FAL)
C      GO TO 11
C
C      100 WRITE(6,111)
C      111 FORMAT( /,5X, '***** THE APPLIED LOAD IS TOO HIGH    *****/')
C
C      11 STOP
C
C      END
C
*OPTIONS IN EFFECT*  ID,EBCDIC,SOURCE,NOLIST,NUDECK,LOAD,NOMAP
*OPTIONS IN EFFECT*  NAME = MAIN      LINECNT = 57
*STATISTICS*  SOURCE STATEMENTS = 32,PROGRAM SIZE = 798
*STATISTICS*  NO DIAGNOSTICS GENERATED

```

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8)

```

      SUBROUTINE PGAUSS
      IMPLICIT REAL*8 (A-H,O-Z)
      COMMON /ACODR/ X(400),Y(400),RG(2),SG(2),WG(2)
      1           SI(4),RI(4),P(20),Q(20)
      COMMON /ANOD/ NX,NP,NEQ,NBC(37),NFIIX(37),LINT,NBAND
      LINT=2
      RG(1)=0.577350269187
      RG(2)=-RG(1)
      SG(1)=RG(1)
      SG(2)=RG(2)
      C
      WG(1)=1.
      WG(2)=1.
      C
      RI(1)=-1.0
      SI(1)=-1.0
      RI(2)=-1.0
      SI(2)= 1.0
      RI(3)= 1.0
      SI(3)=-1.0
      RI(4)= 1.0
      SI(4)= 1.0
      C
      RETURN
      END
      *OPTIONS IN EFFECT*  ID.EBCDIC SOURCE.NOLIST.NODECK,LOAD.NOMAP
      *OPTIONS IN EFFECT*  NAME = PGAUSS  LINECNT = 57
      *STATISTICS*  SOURCE STATEMENTS = 21,PROGRAM SIZE = 424
      *STATISTICS*  NO DIAGNOSTICS GENERATED

```

MICHIGAN TERMINAL SYSTEM FORTRAN G(21 8)

MAIN 06-01-82 16:14:21 PAGE 001

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8) MAIN 06-01-82 16:14:21 PAGE P002

C 4.2) WD : WIDTH RATIO W/D 148.000
 C 4.3) ED : EDGE RATIO E/D 149.000
 C 4.4) LD : LENGTH RATIO L/D 150.000
 C 4.5) NLY : A HALF NUMBER OF TOTAL PLIES IN THE
 SYMMETRIC LAMINATE 151.000
 C 4.6) HI : ONE HALF OF LAMINATE THICKNESS 152.000
 C 4.7) THICK(I) : THICKNESS OF I-TH PLY (NRAN=1) 153.000
 C 154.000
 C 155.000
 C 156.000
 C 157.000
 C 158.000
 C 159.000
 C 160.000
 C 161.000
 C 162.000
 C 163.000
 C 164.000
 C 165.000
 C 166.000
 C 167.000
 C 168.000
 C 169.000
 C 170.000
 C 171.000
 C 171.000
 C 171.000
 C 172.000
 C 173.000
 C 174.000
 C 175.000
 C 176.000
 C 177.000
 C 178.000
 C 179.000
 C 180.000
 C 181.000
 C 182.000
 C 183.000
 C 184.000
 C 185.000
 C 186.000
 C 187.000
 C 188.000
 C 189.000
 C 190.000
 C 191.000
 C 192.000
 C 193.000
 C 194.000
 C 195.000
 C 196.000
 C 197.000
 C 198.000
 C 199.000

* NOTE :
 IF NRAN=0. NLY. THICK(I) AND ANT(I) DO NOT
 HAVE TO BE INPUT.

IF NRAN=1. INPUT ANG=0.0 AND DANG=1.0.

0002 IMPLICIT REAL*8 (A-H,O-Z)
 0003 COMMON /ACONT/ ANG,DANG,NLY,ITEST,XX,SS,RT,RC,HI,ANT(9),THICK(9),N
 1RAN
 0004 COMMON /AMAT/ E1,E2,V12,G12,C(3,3),QA(40,3,3)
 0005 COMMON /ADIM/ D,WD,ED,ALD,W,E,AL
 0006 COMMON /AFORE/ PF,DP,F(706)
 0007 COMMON /ANOD/ NX,NP,NEQ,NBC(37),NFIIX(37),LINT,NBAND
 0008 COMMON /ALEM/ NODXY,NELX,IJK(4,400)
 0009 COMMON /ACODR/ X(400),Y(400),RG(2),SG(2),WG(2),
 1 SI(4),RI(4),P(20),Q(20)
 READ (5,1) ITEST,NRAN
 1 FORMAT (I5)
 0011 READ (5,10) ANG,DANG
 0012 READ (5,10) E1,V12,G12
 0013 READ (5,10) D,WD,ED,ALD
 0014 READ (5,10) XX,SS,RT,RC
 0015 READ (5,10) HI
 0016 C
 IF (NRAN .EQ. 0) GO TO 6
 READ (5,1) NLY
 READ (5,10) (ANT(I),THICK(I),I=1,NLY)
 10 FORMAT (4F15.5)
 C 6 CONTINUE
 NP=4

MICHIGAN TERMINAL SYSTEM FORTRAN G(21,8)

INPUT 06-01-82 16:14:21 PAGE P003

```

0023          NX=353      200.000
0024          NELX=306    201.000
0025          NEO=706    202.000
0026          NBAND=82    203.000
0027          E2=100    204.000
0028          PF=500    205.000
                           206.000
                           207.000
                           208.000
                           209.000
                           210.000
                           211.000
                           212.000
                           213.000
                           214.000
                           215.000
                           216.000
                           217.000
                           218.000
                           219.000
                           220.000
                           221.000
                           222.000
                           223.000
                           224.000
                           225.000
                           226.000
                           227.000
                           228.000
                           229.000
                           230.000
                           231.000
                           232.000
                           233.000
                           234.000
                           235.000
                           236.000
                           237.000
                           238.000
                           239.000
                           240.000
                           241.000
                           242.000
                           243.000
                           244.000
                           245.000
                           246.000
                           247.000
                           248.000
                           249.000
                           250.000
                           251.000
                           252.000
                           253.000
                           254.000

0029          C
0030          C
0031          C
0032          C
0033          C
0034          C
0035          C
0036          C
0037          C
0038          C
0039          C
0040          C
0041          C
0042          C
0043          C
0044          C
0045          C
0046          C
0047          C
0048          C
0049          C
0050          C
0051          C
0052          C
0053          C
0054          C
0055          C
0056          C
0057          C
0058          C
0059          C
0060          C
0061          C
0062          C
0063          C
0064          C
0065          C
0066          C
0067          C
0068          C

          W=WD*D
          E=ED*D
          AL=ALD*D
          TP=4.0*PF*HI
          CALL AMESH
          **** INPUT FIXED BOUNDARY POINTS ****
          NEC(1)=353
          NBC(2)=352
          NBC(3)=351
          NBC(4)=350
          NBC(5)=349
          NBC(6)=348
          NBC(7)=347
          NBC(8)=346
          NBC(9)=345
          NBC(10)=344
          NBC(11)=343
          NBC(12)=342
          NBC(13)=341
          NBC(14)=340
          NBC(15)=339
          NBC(16)=1
          NBC(17)=2
          NBC(18)=3
          NBC(19)=4
          NBC(20)=5
          NBC(21)=6
          NBC(22)=7
          NBC(23)=8
          NBC(24)=9
          NBC(25)=10
          NBC(26)=11
          IF(I TEST .EQ. 0) GO TO 40
          NBC(27)=338
          NBC(28)=323
          NBC(29)=308
          NBC(30)=290
          NBC(31)=291
          NBC(32)=292
          NBC(33)=293
          IF(I TEST .EQ. 1) GO TO 50

```

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8)

```

0069      40 NBC(27)=22          INPUT      06-01-82      16:14:21      PAGE  P004
0070      NBC(28)=33
0071      NBC(29)=44
0072      NBC(30)=55
0073      NBC(31)=66
0074      NBC(32)=77
0075      NBC(33)=88
0076      NBC(34)=99
0077      NBC(35)=103
0078      NBC(36)=107
0079      NBC(37)=111

          C      50 NFIX(1)=11
          C      IF(ITEST .EQ. 0) NFIX(1)=10
          C      NFIX(2)=10
          C      NFIX(3)=10
          C      NFIX(4)=10
          C      NFIX(5)=10
          C      NFIX(6)=10
          C      NFIX(7)=10
          C      NFIX(8)=10
          C      NFIX(9)=10
          C      NFIX(10)=10
          C      NFIX(11)=10
          C      NFIX(12)=10
          C      NFIX(13)=10
          C      NFIX(14)=10
          C      NFIX(15)=10
          C      NFIX(16)=10

          C      IF(ITEST .EQ. 1) NFIX(16)=11
          C      NFIX(17)=10
          C      NFIX(18)=10
          C      NFIX(19)=10
          C      NFIX(20)=10
          C      NFIX(21)=10
          C      NFIX(22)=10
          C      NFIX(23)=10
          C      NFIX(24)=10
          C      NFIX(25)=10
          C      NFIX(26)=11

          C      IF(ITEST .EQ. 1) NFIX(26)=10
          C      NFIX(27)=1
          C      NFIX(28)=1
          C      NFIX(29)=1
          C      NFIX(30)=1
          C      NFIX(31)=1
          C      NFIX(32)=1
          C      NFIX(33)=1
          C      IF(ITEST .EQ. 1) GO TO 55

```

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8)

```

0116      NFIX(34)=1          310.000
0117      NFIX(35)=1          311.000
0118      NFIX(36)=1          312.000
0119      NFIX(37)=1          313.000
0120      C      ***** INPUT NODAL UNIT FORCES *****
0121      C      55  FT=X(262)/(W/2.0)          314.000
0122      FT3=X(316)/(W/2.0)          315.000
0123      FT2=FT-FT3
0124      FT1=1-FT
0125      C      DO 100 I=1,706          316.000
0126      F(I)=0.0          317.000
0127      100  CONTINUE          318.000
0128      IF(ITEST.EQ.1) GO TO 155          319.000
0129      F(586)=-FT1/6.0          320.000
0130      F(584)=-FT1/3.0          321.000
0131      F(582)=-FT1/3.0          322.000
0132      F(580)=-(FT1/6.0)-(FT2/4.0)          323.000
0133      F(616)=-(FT2/2.0)          324.000
0134      F(646)=-(FT2/4.0)-(FT3/4.0)          325.000
0135      F(676)=-(FT3/2.0)          326.000
0136      F(706)=-FT3/4.0          327.000
0137      IF(ITEST.EQ.0) GO TO 150          328.000
0138      C      155  PI=3.141596535          329.000
0139      F(1)=2.0*(4.0/(D*PI))*0.0          330.000
0140      F(2)=0.062499988          331.000
0141      F(23)=0.012193140          332.000
0142      F(24)=0.12319906          333.000
0143      F(45)=0.023917705          334.000
0144      F(46)=0.12024245          335.000
0145      F(67)=0.034723127          336.000
0146      F(68)=0.11446683          337.000
0147      F(89)=0.04194158          338.000
0148      F(90)=0.10665416          339.000
0149      F(111)=0.051966834          340.000
0150      F(112)=0.09723130          341.000
0151      F(133)=0.057742454          342.000
0152      F(134)=0.086417711          343.000
0153      F(155)=0.061299064          344.000
0154      F(156)=0.074693147          345.000
0155      F(177)=0.062499988          346.000
0156      F(178)=0.062500007          347.000
0157      F(223)=0.061299072          348.000
0158      F(224)=0.050306867          349.000
0159      F(246)=0.038582300          350.000
0160      F(267)=0.051966855          351.000
0161      F(268)=0.027776877          352.000
0162      F(289)=0.044194186          353.000
0163      F(290)=0.018305843          354.000
0164      F(311)=0.034723159          355.000

```

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8)

	INPUT	06-01-82	16:14:21	PAGE P006
O165	F(312)=0.010533165			365.000
O166	F(333)=0.023917741			366.000
O167	F(334)=0.0047575414			367.000
O168	F(355)=0.012193179			368.000
O169	F(356)=0.0012009269			369.000
O170	F(377)=0.0020453255			370.000
O171	F(378)=2.0*0.00			371.000
	C	372.000		
	C	373.000		
	C	374.000		
	C	375.000		
O172	150 CONTINUE	376.000		
O173	RETURN	377.000		
O174	END			
	OPTIONS IN EFFECT ID,EBCDIC,SOURCE,NOLIST,NODECK,LOAD,NOMAP			
	OPTIONS IN EFFECT NAME = INPUT LINECNT = 57			
	STATISTICS SOURCE STATEMENTS = 174,PROGRAM SIZE = 2894			
	STATISTICS NO DIAGNOSTICS GENERATED			

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8)

06-01-82 16:14:34 PAGE P001

AMESH MICHIGAN TERMINAL SYSTEM FORTRAN G(21 8)

MICHIGAN TERMINAL SYSTEM FORTRAN G (21.8)

06-01-82

16:14:34

AMESH

PAGE 0003

```

0091      BX(6,8)=BX(3,6)
          BY(6,8)=BY(3,6)
          C
0093      C      BX(7,1)=D/2.+ZN
          BY(7,1)=((BY(6,4)-BY(6,3))/4.)*3.+BY(6,3)
          BX(7,2)=BX(6,3)
          BY(7,2)=BY(6,3)
          BX(7,3)=W/2.0
          BY(7,3)=BY(6,3)
          BX(7,4)=W/2.0
          BY(7,4)=BY(7,1)
          C      BX(8,1)=BX(6,2)
          BY(8,1)=BY(6,2)
          BX(8,5)=D/2.*DSIN(TH14)
          BY(8,5)=D/2.*DCOS(TH14)
          BX(8,2)=D/2.
          BY(8,2)=0.0
          BX(8,6)=D/2.+ZN/2.0
          BY(8,6)=0.0
          BX(8,3)=(D/2.)*ZN
          BY(8,3)=0.0
          BX(8,7)=(D/2.)*ZN
          BY(8,7)=BX(8,7)/DCOS(TH2)
          BX(8,4)=BX(6,3)
          BY(8,4)=BY(6,3)
          BX(8,8)=BX(6,6)
          BY(8,8)=BY(6,6)
          C      BX(9,1)=BX(8,4)
          BY(9,1)=BY(8,4)
          BX(9,2)=BX(8,3)
          BY(9,2)=BY(8,3)
          BX(9,3)=W/2.0
          BY(9,3)=0.0
          BX(9,4)=W/2.0
          BY(9,4)=BY(8,4)
          BY(9,6)=BY(8,6)
          C      BX(10,1)=D/2.
          BY(10,1)=0.0
          BX(10,5)=BX(8,5)
          BY(10,5)=-1*BY(8,5)
          BX(10,2)=BX(8,1)
          BY(10,2)=-BY(8,1)
          BX(10,6)=BX(8,8)
          BY(10,6)=-BY(8,8)
          BX(10,3)=BX(8,4)
          BY(10,3)=-BY(8,4)
          BX(10,7)=BX(8,7)
          BY(10,7)=-BY(8,7)
          BX(10,4)=BX(8,3)

```

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8)

AMESH

PAGE P004

```

0138          BY(10,4)=BY(8,3)          541 000
0139          BX(10,8)=BX(8,6)          542 000
0140          BY(10,8)=-BY(8,6)          543 000
0141          C
0142          BX(11,1)=BX(9,2)          544 000
0143          BY(11,1)=-BY(9,2)          545 000
0144          BX(11,2)=BX(9,1)          546 000
0145          BY(11,2)=-BY(9,1)          547 000
0146          BX(11,3)=BX(9,4)          548 000
0147          BY(11,3)=-BY(9,4)          549 000
0148          BX(11,4)=BX(9,3)          550 000
0149          BY(11,4)=-BY(9,3)          551 000
0150          C
0151          BX(12,1)=BX(6,2)          552 000
0152          BY(12,1)=-BY(6,2)          553 000
0153          BX(12,5)=BX(6,5)          554 000
0154          BY(12,5)=-BY(6,5)          555 000
0155          BX(12,2)=BX(6,1)          556 000
0156          BY(12,2)=-BY(6,1)          557 000
0157          BX(12,6)=BX(6,8)          558 000
0158          BY(12,6)=-BY(6,8)          559 000
0159          BX(12,3)=BX(6,4)          560 000
0160          BY(12,3)=-BY(6,4)          561 000
0161          BX(12,7)=BX(6,7)          562 000
0162          BY(12,7)=-BY(6,7)          563 000
0163          BX(12,4)=BX(6,3)          564 000
0164          BY(12,4)=-BY(6,3)          565 000
0165          BX(12,8)=BX(6,6)          566 000
0166          BY(12,8)=-BY(6,6)          567 000
0167          C
0168          BX(13,1)=BX(7,2)          568 000
0169          BY(13,1)=-BY(7,2)          569 000
0170          BX(13,2)=BX(6,4)          570 000
0171          BY(13,2)=-BY(6,4)          571 000
0172          BX(13,3)=BX(6,3)          572 000
0173          BY(13,3)=-BY(6,3)          573 000
0174          C
0175          BX(14,1)=BX(13,2)          574 000
0176          BY(14,1)=-BY(13,2)          575 000
0177          BX(14,2)=BX(13,2)          576 000
0178          BY(14,2)=-(AL-E)          577 000
0179          BX(14,3)=W/2.0          578 000
0180          BY(14,3)=BY(14,2)          579 000
0181          BX(15,1)=BX(3,2)          580 000
0182          BY(15,1)=-BY(3,2)          581 000
0183          C

```

MICHIGAN TERMINAL SYSTEM FORTRAN G(21,8)

PAGE 005

06-01-82

16:14:34

AMESH

```

BX(15,5)=BX(3,5)
BY(15,5)=-BY(3,5)
BX(15,2)=BX(3,1)
BY(15,2)=-BY(3,1)
BX(15,6)=BX(3,8)
BY(15,6)=-BY(3,8)
BX(15,3)=BX(3,4)
BY(15,3)=-BY(3,4)
BX(15,7)=BX(3,7)
BY(15,7)=-BY(3,7)
BX(15,4)=BX(3,3)
BY(15,4)=-BY(3,3)
BX(15,8)=BX(3,6)
BY(15,8)=-BY(3,6)

C      C      BX(16,1)=(BX(15,4)-BX(15,3))/2.+BX(15,3)
BY(16,1)=-(D/2.+ZN)
BX(16,2)=BX(15,3)
BY(16,2)=-(D/2.+ZN)
BX(16,3)=BX(15,3)
BY(16,3)=-(AL-E)
BX(16,4)=BX(16,1)
BY(16,4)=-(AL-E)

C      C      BX(17,1)=BX(1,2)
BY(17,1)=-BY(1,2)
BX(17,5)=BX(1,5)
BY(17,5)=-BY(1,5)
BX(17,2)=BX(1,1)
BY(17,2)=-BY(1,1)
BX(17,6)=BX(1,8)
BY(17,6)=-BY(1,8)
BX(17,3)=BX(1,4)
BY(17,3)=-BY(1,4)
BX(17,7)=BX(1,7)
BY(17,7)=-BY(1,7)
BX(17,4)=BX(1,3)
BY(17,4)=-BY(1,3)
BX(17,8)=BX(1,6)
BY(17,8)=-BY(1,6)

C      C      BX(18,1)=BX(17,4)
BY(18,1)=BY(17,4)
BX(18,2)=O.O
BY(18,2)=BY(15,4)
BX(18,3)=O.O
BY(18,3)=-(AL-E)
BX(18,4)=BX(17,4)
BY(18,4)=-(AL-E)
NBLOCK=18
NES(1)=0
0229
0230

```

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8)

AMESH

PAGE P006

16:14:34

```

0231      NES(2)=28
0232      NES(3)=40
0233      NES(4)=68
0234      NES(5)=80
0235      NES(6)=89
0236      NES(7)=117
0237      NES(8)=126
0238      NES(9)=154
0239      NES(10)=166
0240      NES(11)=194
          NS(11)=206
          NS(12)=220
          NS(13)=226
0241      NES(14)=226
0242      NES(15)=247
          NS(16)=261
          NS(17)=268
          NS(18)=282
          C
          C
0248      NS(1)=0
0249      NS(2)=7
          NS(3)=44
0250      NS(4)=51
0251      NS(5)=95
0252      NS(6)=88
0253      NS(7)=118
0254      NS(8)=144
0255      NS(9)=151
0256      NS(10)=188
0257      NS(11)=195
0258      NS(12)=232
0259      NS(13)=239
0260      NS(14)=261
0261      NS(15)=254
0262      NS(16)=300
0263      NS(17)=308
          NS(18)=315
          C
          C
0266      DO 11 I=1,18
0267      11 K12(I)=0
          C
          C
0268      K12(2)=7
0269      K12(4)=7
0270      K12(7)=7
0271      K12(9)=7
          K12(11)=7
0272      K12(13)=7
0273      K12(15)=7
0274      K12(16)=7
          K12(18)=7
          C
          C
0276      DO 12 I=1,18
0277      12 K43(I)=0
          C

```

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8)

AMESH 06-01-82 16:14:34 PAGE P007

```

0278      K43(1)=3      706.000
0279      K43(3)=3      707.000
0280      K43(6)=3      708.000
0281      K43(8)=3      709.000
0282      K43(10)=3     710.000
0283      K43(12)=3     711.000
0284      K43(15)=7     712.000
0285      K43(17)=7     713.000
0286      C              714.000
0287      C              715.000
0288      C              716.000
0289      C              717.000
0290      C              718.000
0291      DO 14 I=1,18
0292      14 MX(I)=4
0293      MX(5)=3      719.000
0294      MX(7)=3      720.000
0295      MX(12)=2     721.000
0296      MX(13)=2     722.000
0297      MX(14)=7     723.000
0298      MX(15)=2     724.000
0299      MX(16)=1     725.000
0300      MX(17)=2     726.000
0301      MX(18)=2     727.000
0302      DO 15 I=1,18
0303      15 MY(I)=7
0304      MY(2)=3      728.000
0305      MY(4)=3      729.000
0306      MY(5)=3      730.000
0307      MY(7)=3      731.000
0308      MY(9)=3      732.000
0309      MY(11)=3     733.000
0310      MY(13)=3     734.000
0311      MY(14)=3     735.000
0312      C              736.000
0313      C              737.000
0314      C              738.000
0315      C              739.000
0316      C              740.000
0317      C              741.000
0318      C              742.000
0319      C              743.000
0320      C              744.000
0321      C              745.000
0322      C              746.000
0323      C              747.000
0324      C              748.000
0325      C              749.000
0326      C              750.000
0327      C              751.000
0328      C              752.000
0329      C              753.000
0330      C              754.000
0331      C              755.000
0332      C              756.000
0333      C              757.000
0334      C              758.000
0335      C              759.000
0336      C              760.000

```

MICHIGAN TERMINAL SYSTEM FORTRAN G(21,8)

AMESH 06-01-82 16:14:34 PAGE P008

```

0317      KY(5)=1          761.000
0318      KY(7)=1          762.000
0319      KY(9)=1          763.000
0320      KY(11)=1          764.000
0321      KY(13)=1          765.000
0322      KY(14)=1          766.000
0323      KY(16)=1          767.000
0324      KY(18)=1          768.000
0325      C
0326      C      DO 17 I=1,18          769.000
17      KSC(I)=4          770.000
0327      C      KSC(1)=8          771.000
0328      C      KSC(3)=8          772.000
0329      C      KSC(6)=8          773.000
0330      C      KSC(8)=8          774.000
0331      C      KSC(10)=8         775.000
0332      C      KSC(12)=8         776.000
0333      C      KSC(15)=8         777.000
0334      C      KSC(17)=8         778.000
0335      C      DO 300 I=1,NBLOCK
300     C      *****COORDINATE *****
301     C
302     C
303     C
304     C
305     C
306     C
307     C
308     C
309     C
310     C
311     C
312     C
313     C
314     C
315     C
316     C
317     C
318     C
319     C
320     C
321     C
322     C
323     C
324     C
325     C
326     C
327     C
328     C
329     C
330     C
331     C
332     C
333     C
334     C
335     C
336     C
337     C
338     C
339     C
340     C
341     C
342     C
343     C
344     C
345     C
346     C
347     C
348     C
349     C
350     C
351     C
352     C
353     C
354     C
355     C
356     C
357     C
358     C
359     C
0336      MXX=MX(I)
0337      MYY=MY(I)
0338      MXX1=MXX+1
0339      MYY1=MYY+1
0340      DO 100 IX=1,MXX1
100     IF(IX.EQ.1) NSI=NS(I)
101     IF(IX.GT.1.AND.IX.LT.MXX1) NSI=NS(I)+K4(I)+(IX-1)*(MYY1+K12(I)+K43(I))
102     IF(IX.EQ.MXX1) NSI=NS(I)+K4(I)+K2+(IX-1)*(MYY1+K12(I)+K43(I))
103     XLI=FUN1(IX,MXX,KX)
104     DO 100 IY=1,MYY1
105     KY1=KY(I)
106     YLI=FUN1(IY,MYY,KY1)
107     CONTINUE
108     IF(KSC(I).EQ.4) CALL SEREND(SH,XLI,YLI,4)
109     IF(KSC(I).EQ.8) CALL SEREND(SH,XLI,YLI,8)
110     XI=0.
111     YI=0.
112     KSC1=KSC(I)
113     DO 102 K=1,KSC1
114     SHK=SH(K)
115     XI=XI+BX(I,K)*SHK
116     YI=YI+BY(I,K)*SHK
117     NIXY=NSI+IY
118     X(NIXY)=XI

```

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8)

PAGE P009

06-01-82

16:14:34

```

0360          Y(NIXY)=YI
0361          100 CONTINUE
C
C      ***** ELEMENT *****
C      ***** 4-NODE ELEMENT *****
C
0362          DO 202 IX=1,MXX
0363          NSI=NS(I)+(IX-1)*(MYY1+K12(I)+K43(I))
0364          DO 202 IY=1,MYY
0365          NN=NSI+IY
0366          NEL=NES(I)+(IX-1)*MY(I)+IY
0367          IJK(1,NEL)=NN+K4(I)
0368          IF(IX.EQ.1) IJK(1,NEL)=NN
0369          IJK(3,NEL)=NN+K4(I)+(MYY1+K12(I)+K43(I))
0370          IJK(4,NEL)=IJK(3,NEL)+1
0371          IJK(2,NEL)=IJK(1,NEL)+1
0372          IF(IX.NE.MXX) GO TO 202
0373          IJK(2,NEL)=IJK(2,NEL)+K2
0374          IJK(3,NEL)=IJK(3,NEL)+K2
0375          202 CONTINUE
C
0376          206 CONTINUE
C
0377          300 CONTINUE
0378          IJK(1,297)=96
0379          IJK(2,297)=100
0380          IJK(3,297)=119
0381          IJK(4,297)=120
0382          IJK(1,298)=100
0383          IJK(2,298)=104
0384          IJK(3,298)=120
0385          IJK(4,298)=121
0386          IJK(1,299)=104
0387          IJK(2,299)=108
0388          IJK(3,299)=121
0389          IJK(4,299)=122
0390          IJK(1,300)=122
0391          IJK(2,300)=266
0392          IJK(3,300)=301
0393          IJK(4,300)=302
0394          IJK(1,301)=266
0395          IJK(2,301)=270
0396          IJK(3,301)=302
0397          IJK(4,301)=303
0398          IJK(1,302)=270
0399          IJK(2,302)=274
0400          IJK(3,302)=303
0401          IJK(4,302)=304
0402          IJK(1,303)=274
0403          IJK(2,303)=278
0404          IJK(3,303)=304
0405          IJK(4,303)=305
0406          IJK(1,304)=278
0407          IJK(2,304)=282

```

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8)

```

      AMESH          06-01-82      16: 14: 34      PAGE P010
      IJK(3,304)=305
      IJK(4,304)=306
      IJK(1,305)=282
      IJK(2,305)=286
      IJK(3,305)=306
      IJK(4,305)=307
      IJK(1,306)=286
      IJK(2,306)=290
      IJK(3,306)=307
      IJK(4,306)=308
      WRITE (6,900)
      C 900 FORMAT(//5X,'<COORDINATE>',/)
      C WRITE(6,902) (I,X(I),Y(I),I=1,NX)
      C 902 FORMAT(10X,15.2X,2F10.3,5X,15.2X,2F10.3)
      C WRITE(6,904)
      C 904 FORMAT(//5X,'<ELEMENT>',/)
      C DO 906 NEL=1,NELX
      C 906 WRITE(6,908) NEL,(IJK(I,NEL),I=1,NP)
      C 908 FORMAT(10X,15.5X,9I5)
      C WRITE(6,909)(P(I),Q(I),LEB(I),I=1,18)
      C 909 FORMAT(//.5X.,P=,.E15.6,5X.,Q=,.E15.6,5X.,LEB=,.I3/)
      RETURN
      END
      *OPTIONS IN EFFECT*  ID,EBCDIC, SOURCE, NOLIST, NODECK, LOAD, NOMAP
      *OPTIONS IN EFFECT*  NAME = AMESH      LINECNT = 57
      *STATISTICS*  SOURCE STATEMENTS = 419, PROGRAM SIZE = 13950
      *STATISTICS*  NO DIAGNOSTICS GENERATED

```

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8)

		SEREND	06-01-82	16:14:56	PAGE 001
0001	C	SUBROUTINE SEREND(SH,XL,YL,KKK)			
0002	C	IMPLICIT REAL*8(A-H,O-Z)			
0003	C	DIMENSION SH(8)			
0004	C	K4=KKK/4	893.000	893.000	
0005	C	GO TO (100,200),K4	894.000	894.000	
	C	*** 4-NODE ELEMENT *****	895.000	895.000	
0006	C	100 CONTINUE	896.000	896.000	
0007		SH(1)=0.25*(1-XL)*(1,-YL)	897.000	897.000	
0008		SH(2)=0.25*(1.+XL)*(1,-YL)	898.000	898.000	
0009		SH(3)=0.25*(1.+XL)*(1.+YL)	899.000	899.000	
0010		SH(4)=0.25*(1.-XL)*(1.+YL)	900.000	900.000	
0011		RETURN	901.000	901.000	
	C	*** 8-NODE ELEMENT *****	902.000	902.000	
0012	C	200 CONTINUE	903.000	903.000	
0013		SH(1)=-0.25*(XL+YL+1.)*(XL-1.)*(YL-1.)	904.000	904.000	
0014		SH(2)=-0.25*(XL-YL-1.)*(XL+1.)*(YL-1.)	905.000	905.000	
0015		SH(3)= 0.25*(XL+YL-1.)*(XL+1.)*(YL+1.)	906.000	906.000	
0016		SH(4)= 0.25*(XL-YL+1.)*(XL-1.)*(YL+1.)	907.000	907.000	
0017		SH(5)= 0.5*(XL*XL-1.)*(YL-1.)	908.000	908.000	
0018		SH(6)=-0.5*(XL+1.)*(YL*YL-1.)	909.000	909.000	
0019		SH(7)=-0.5*(XL*XL-1.)*(YL+1.)	910.000	910.000	
0020		SH(8)= 0.5*(XL-1.)*(YL*YL-1.)	911.000	911.000	
0021		RETURN	912.000	912.000	
0022		END	913.000	913.000	
		OPTIONS IN EFFECT ID,EBCDIC,SOURCE,NOLIST,NODECK,LOAD,NOMAP	914.000	914.000	
		OPTIONS IN EFFECT NAME = SEREND . LINECNT = 57	915.000	915.000	
		STATISTICS SOURCE STATEMENTS = 22, PROGRAM SIZE = 868	916.000	916.000	
		STATISTICS NO DIAGNOSTICS GENERATED	917.000	917.000	
			918.000	918.000	
			919.000	919.000	
			920.000	920.000	

```

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8)          FUN1          06-01-82      16:14:59      PAGE 001
          FUNCTION FUN1(I,M,K)
0001      C
0002      C
0003      IMPLICIT REAL*8(A-H,O-Z)
0004      RI=I
0005      RM=M
0006      GO TO 100,102,104,K
0007      FUN1=-1.+2.* (RI-1.)/RM
0008      RETURN
0009      102 FUN1=-1.+2.* (RI-1.)*RI/(RM*(RM+1.))
0010      RETURN
0011      104 FUN1=-1.+2.* (RI-1.)*(2.*RM-RI+2.)/(RM*(RM+1.))
0012      RETURN
0013      END
*OPTIONS IN EFFECT*  *D EBCDIC, SOURCE, NOLIST, NODECK, LOAD, NOMAP
*OPTIONS IN EFFECT*  NAME = FUN1      , LINECNT = 57
*STATISTICS*  SOURCE STATEMENTS = 12, PROGRAM SIZE = 634
*STATISTICS*  NO DIAGNOSTICS GENERATED

```

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8) FTH 06-01-82 16:14:59 PAGE P001
 0001 C FUNCTION FTH(TH) 935.000
 C 936.000
 0002 C IMPLICIT REAL *8(A-H,O-Z) 937.000
 COMMON /ADIM/ 9.0D.0,ALD,W,E,AL 938.000
 0003 C 939.000
 0004 C 940.000
 Z=W/3.6 941.000
 0005 C 942.000
 ZN=Z-(D/2.0) 943.000
 0006 C 944.000
 FTH=((((D/2.0)+ZN)/DCOS(TH))-(D/2.0))/2.+(D/2.0))
 0007 RETURN
 0008 END
 OPTIONS IN EFFECT ID EBCDIC, SOURCE, NOLIST, NODECK, LOAD, NOMAP
 OPTIONS IN EFFECT NAME = FTH LINECNT = 57
 STATISTICS SOURCE STATEMENTS = 8,PROGRAM SIZE = 442
 STATISTICS NO DIAGNOSTICS GENERATED

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8)

POINT 06-01-82 16:15:00 PAGE 001

```

0001      SUBROUTINE POINT(LEB)
0002      IMPLICIT REAL*8(A-H,O-Z)
0003      COMMON /ACONT/ ANG,DANG,NLY,ITEST,XX,SS,RT,RC,HI,ANT(9),THICK(9),N
1RAN
0004      COMMON /ADIM/ D,WD,ED,ALD,W,E,AL
0005      COMMON /ALEM/ NODXY,NELX,IJK(4,400)
0006      COMMON /ACOOR/ X(400),Y(400),RG(2),SG(2),WG(2)
1      COMMON /SI/ (4),RI(4),P(20),Q(20)
1      DIMENSION LEB(20)
0007      C
0008      RLC=RC+(D/2.0)
0009      RL=RT+(D/2.0)
0010      CA=0.09817477
0011      PI=3.141596535
0012      P(1)=(RL+(RC-RT)*DCOS(CA/6.0))*DSIN(CA/6.0)
0013      Q(1)=(RL+(RC-RT)*DCOS(CA/6.0))*DCOS(CA/6.0)
0014      P(2)=(RL+(RC-RT)*DCOS(CA/2.0))*DSIN(CA/2.0)
0015      Q(2)=(RL+(RC-RT)*DCOS(CA/2.0))*DCOS(CA/2.0)
0016      P(3)=(RL+(RC-RT)*DCOS(3*CA/2.0))*DSIN(3*CA/2.0)
0017      Q(3)=(RL+(RC-RT)*DCOS(3*CA/2.0))*DCOS(3*CA/2.0)
0018      P(4)=(RL+(RC-RT)*DCOS(5*CA/2.0))*DSIN(5*CA/2.0)
0019      Q(4)=(RL+(RC-RT)*DCOS(5*CA/2.0))*DCOS(5*CA/2.0)
0020      P(5)=(RL+(RC-RT)*DCOS(7*CA/2.0))*DSIN(7*CA/2.0)
0021      Q(5)=(RL+(RC-RT)*DCOS(7*CA/2.0))*DCOS(7*CA/2.0)
0022      P(6)=(RL+(RC-RT)*DCOS(9*CA/2.0))*DSIN(9*CA/2.0)
0023      Q(6)=(RL+(RC-RT)*DCOS(9*CA/2.0))*DCOS(9*CA/2.0)
0024      P(7)=(RL+(RC-RT)*DCOS(11*CA/2.0))*DSIN(11*CA/2.0)
0025      Q(7)=(RL+(RC-RT)*DCOS(11*CA/2.0))*DCOS(11*CA/2.0)
0026      P(8)=(RL+(RC-RT)*DCOS(13*CA/2.0))*DSIN(13*CA/2.0)
0027      Q(8)=(RL+(RC-RT)*DCOS(13*CA/2.0))*DCOS(13*CA/2.0)
0028      P(9)=(RL+(RC-RT)*DCOS(15*CA/2.0))*DSIN(15*CA/2.0)
0029      Q(9)=(RL+(RC-RT)*DCOS(15*CA/2.0))*DCOS(15*CA/2.0)
0030      P(10)=(RL+(RC-RT)*DCOS(17*CA/2.0))*DSIN(17*CA/2.0)
0031      Q(10)=(RL+(RC-RT)*DCOS(17*CA/2.0))*DCOS(17*CA/2.0)
0032      P(11)=(RL+(RC-RT)*DCOS(19*CA/2.0))*DSIN(19*CA/2.0)
0033      Q(11)=(RL+(RC-RT)*DCOS(19*CA/2.0))*DCOS(19*CA/2.0)
0034      P(12)=(RL+(RC-RT)*DCOS(21*CA/2.0))*DSIN(21*CA/2.0)
0035      Q(12)=(RL+(RC-RT)*DCOS(21*CA/2.0))*DCOS(21*CA/2.0)
0036      P(13)=(RL+(RC-RT)*DCOS(23*CA/2.0))*DSIN(23*CA/2.0)
0037      Q(13)=(RL+(RC-RT)*DCOS(23*CA/2.0))*DCOS(23*CA/2.0)
0038      P(14)=(RL+(RC-RT)*DCOS(25*CA/2.0))*DSIN(25*CA/2.0)
0039      Q(14)=(RL+(RC-RT)*DCOS(25*CA/2.0))*DCOS(25*CA/2.0)
0040      P(15)=(RL+(RC-RT)*DCOS(27*CA/2.0))*DSIN(27*CA/2.0)
0041      Q(15)=(RL+(RC-RT)*DCOS(27*CA/2.0))*DCOS(27*CA/2.0)
0042      P(16)=(RL+(RC-RT)*DCOS(29*CA/2.0))*DSIN(29*CA/2.0)
0043      Q(16)=(RL+(RC-RT)*DCOS(29*CA/2.0))*DCOS(29*CA/2.0)
0044      P(17)=(RL+(RC-RT)*DCOS(31*CA/2.0))*DSIN(31*CA/2.0)
0045      Q(17)=(RL+(RC-RT)*DCOS(31*CA/2.0))*DCOS(31*CA/2.0)
0046      P(18)=(RL+(RC-RT)*DSIN(CA/6.0))*DCOS(CA/6.0)
0047      Q(18)=(RL+(RC-RT)*DSIN(CA/6.0))*DSIN(CA/6.0)
1      I=0
1      I=I+1
1      IF(I.GT.8) GO TO 110
1      RLX=RLC-Y(I)

```

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8)

PAGE 0002

16:15:00

06-01-82

POINT

```

      IF(RLX .GT. 0.0) GO TO 5      POINT      06-01-82      16:15:00
      LEB(1)=I-1
      LEB(2)=I-1
      K=1
      LI=0
      50 K=K+1
      IF(K .GT. 16) GO TO 100
      KP=7
      IF(K .EQ. 5) KP=19
      IF(K .EQ. 9) KP=28
      IF(K .EQ. 13) KP=16
      C
      M=LEB(K)+KP+LI
      LI=0
      II=IJK(1,M)
      KK=IJK(3,M)
      K1=K+1
      40 IF(X(II) .EQ. X(KK)) TT=P(K1)-X(KK)
      IF(X(II) .EQ. X(KK)) GO TO 45
      TT=(-P(K1)-X(II))*(Y(II)-Y(KK))/(X(II)-X(KK))-Y(II)+Q(K1)
      45 IF(TT .GT. 0.0) GO TO 10
      LEB(K+1)=M-1
      II=IJK(1,M-1)
      KK=IJK(3,M-1)
      M=LEB(K+1)
      LI=LI+1
      GO TO 40
      10 LEB(K1)=M
      GO TO 50
      100 LEB(K+1)=M
      110 RETURN
      END

*OPTIONS IN EFFECT* ID.EBCDIC,SOURCE,NOLIST,NODECK,LOAD,NOMAP
*OPTIONS IN EFFECT* NAME = POINT      LINECNT = 57
*STATISTICS* SOURCE STATEMENTS = 82,PROGRAM SIZE = 5002
*STATISTICS* NO DIAGNOSTICS GENERATED

```

```

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8)          MATRL      06-01-82      16 : 15 : 04      PAGE  P001
0001      SUBROUTINE  MATRL
0002      IMPLICIT REAL*8(A-H,O-Z)
0003      COMMON /ACONT/  ANG,DANG,NLV,ITEST,XX,SS,RT,RC,HI,ANT(9),THICK(9),N
1RAN
0004      COMMON /AMAT/  E1,E2,V12,G12,C(3,3),QA(40,3,3)
0005      COMMON /AFUN/  SHP(3,4),T(40,3,3),STRSS(40,18,3),STR(18,3),ST(11,3
1)
1)
C
C
0006      IF(NRAN .EQ. 0)  NLY=2.0*ANG/DANG+1
0007      V2=E2*V12/E1
0008      DIV=1-V12*V21
0009      Q22=E2/DIV
0010      Q11=E/DIV
0011      Q12=V12*E2/DIV
0012      Q66=G12
C
C
0013      *** COMPUTE INVARIANTI PROPERTIES
0014      U1=(3.*Q11+3.*Q22+2.*Q12+4.*Q66)/8.
0015      U2=(Q11-Q22)/2.
0016      U3=(Q11+Q22-2.*Q12-4.*Q66)/8.0
0017      U4=(Q11+Q22+6.*Q12-4.*Q66)/8.0
0018      U5=(Q11+Q22-2.*Q12+4.*Q66)/8.0
C
C
0019      DO 100 1=1,NLY
1
0020      THTA=90.0-(ANG-DANG*L)
0021      IF(NRAN .EQ. 1) THTA=90.0-ANT(1)
0022      IF(THTA .LT. 180.0) THTA=0.0
0023      DEG=THTA*3.1415926535/180.0
0024      QA(1,1,1)=U1+U2*DCOS(2.*DEG)+U3*DCOS(4.*DEG)
0025      QA(1,1,2)=U4-U3*DCOS(4.*DEG)
0026      QA(1,2,2)=U1-U2*DCOS(2.*DEG)+U3*DCOS(4.*DEG)
0027      QA(1,1,3)=O_5*U2*DSIN(2.*DEG)+U3*DSIN(4.*DEG)
0028      QA(1,2,3)=+O_5*U2*DSIN(2.*DEG)-U3*DSIN(4.*DEG)
0029      QA(1,3,3)=U5-U3*DCOS(4.*DEG)
0030      QA(1,2,1)=QA(1,1,2)
0031      QA(1,3,1)=QA(1,1,3)
0032      QA(1,3,2)=QA(1,2,3)
C
C
0033      *** COMPUTE ROTATION TRANSFORMATION PER PLY *****
C
C
0034      T(1,1,1)=DCOS(DEG)**2
0035      T(1,1,2)=DSIN(DEG)**2
0036      T(1,1,3)=2*DSIN(DEG)*DCOS(DEG)
T(1,2,1)=T(1,1,2)
T(1,2,2)=T(1,1,1)
T(1,2,3)=-T(1,1,3)
0037
0038
0039
0040
0041
C

```

MICHIGAN TERMINAL SYSTEM FORTRAN G(21 8)

0042 100 CONTINUE

0043 C *** COMPUTE LAMINATE PROPERTIES ****

0044 C

0045 DO 200 M=1,3

0046 DO 200 N=1,3

0047 C(M,N)=0.0

0048 DO 150 I=1,NLY

0049 A2=THICK(I)

0050 IF (NRAN .EQ. 0) A2=1.0

0051 150 C(M,N)=C(M,N)+OA(I,M,N)*A2

0052 IF (NRAN .EQ. 0) GO TO 10

0053 C(M,N)=C(M,N)/HI

0054 GO TO 200

0055 10 C(M,N)=C(M,N)/NLY

0056 200 CONTINUE

0057 RETURN

0058 END

*OPTIONS IN EFFECT. ID,EBCDIC, SOURCE, NOLIST, NODECK, LOAD, NOMAP

*OPTIONS IN EFFECT. NAME = MATRL LINECNT = 57

STATISTICS SOURCE STATEMENTS = 56, PROGRAM SIZE = 2044

STATISTICS NO DIAGNOSTICS GENERATED

```

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8)          STIFF          06-01-82      16:15:06      PAGE P001
0001      SUBROUTINE STIFF(MM,A)
0002      C      IMPLICIT REAL*8(A-H,O-Z)
0003      C      COMMON /AMAT/ E1,E2,V12,G12,C(3,3),QA(40,3,3)
0004      C      COMMON /ANOD/ NX,NP,NEQ,NBC(37),NFI(X(37),LINT,NBAND
0005      C      COMMON /ACOOR/ X(400),Y(400),RG(2),SG(2),WG(2)
0006      C      SI(4),RI(4),P(20),Q(20)
0007      C      COMMON /AFUN/ SHP(3,4),T(40,3,3),STRSS(40,18,3),ST(11,3
0008      C      1)
0009      C
0010      C      DIMENSION A(8,8)
0011      C      DO 10 I=1,8
0012      C      DO 10 J=1,8
0013      C      A(I,J)=0.0
0014      C      10 CONTINUE
0015      C      S=SG(M)
0016      C      CALL SHAPEF(MM,XSU,R,S)
0017      C      DV=XSJ*WG(N)*WG(M)
0018      C      *** FOR EACH J NODE COMPUTE CB=C*B   ***
0019      C      DO 100 J=1,NP
0020      C      CB11=C(1,1)*SHP(1,J)*DV+C(1,3)*SHP(2,J)*DV
0021      C      CB12=C(1,2)*SHP(2,J)*DV+C(1,3)*SHP(1,J)*DV
0022      C      CB21=C(1,2)*SHP(1,J)*DV+C(2,3)*SHP(2,J)*DV
0023      C      CB22=C(2,2)*SHP(2,J)*DV+C(2,3)*SHP(1,J)*DV
0024      C      CB31=C(1,3)*SHP(1,J)*DV+C(3,3)*SHP(2,J)*DV
0025      C      CB32=C(2,3)*SHP(2,J)*DV+C(3,3)*SHP(1,J)*DV
0026      C      100 CONTINUE
0027      C      *** FOR EACH I NODE COMPUTE S=BT*CB   ***
0028      C
0029      C      DO 100 I=1,J
0030      C      1=I-1
0031      C      J1=2*I-1
0032      C      I2=2*I
0033      C      J2=2*I
0034      C
0035      C      A(I1,J1)=A(I1,J1)+SHP(1,I)*CB11+SHP(2,I)*CB31
0036      C      A(I1,J2)=A(I1,J2)+SHP(1,I)*CB12+SHP(2,I)*CB32
0037      C      A(I2,J1)=A(I2,J1)+SHP(2,I)*CB21+SHP(1,I)*CB31
0038      C      A(I2,J2)=A(I2,J2)+SHP(2,I)*CB22+SHP(1,I)*CB32
0039      C      100 CONTINUE
0040      C      *** COMPUTE LOWER TRIANGULAR PART BY SYMMETRY
0041      C
0042      C      NL=NP*2
0043      C      DO 200 I=1,NL
0044      C      DO 200 J=1,I
0045      C      200 A(I,J)=A(J,I)
0046      C      RETURN

```

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8)

16 : 15:06 1157.000
PAGE 002
STIFF 06-01-82
OPTIONS IN EFFECT ID, EBCDIC, SOURCE, NOLIST, NODECK, LOAD, NOMAP
OPTIONS IN EFFECT NAME = STIFF LINECNT = 57
STATISTICS SOURCE STATEMENTS = 40, PROGRAM SIZE = 1516
STATISTICS NO DIAGNOSTICS GENERATED
END
00440
MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8)

16:15:06

06-01-82

PAGE 002

78

```

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8)          CFAIL          06-01-82          16:15:07          PAGE  P001
0001          SUBROUTINE CFAIL (KT,MT,FAL)          1158.000
0002          IMPLICIT REAL*8(A-H,O-Z)          1159.000
0003          COMMON /ACONT/ ANG,DANG,NLY,ITEST,XX,SS,RT,RC,HI,ANT(9),THICK(9),N
1RAN          COMMON /AFUN/ SHP(3,4),T(40,3,3),STRSS(40,18,3),STRA(18,3),ST(11,3
1)
1
C
C
C *** FAILURE CRITERIAN ****
C
0005          FAL=0.0          1160.000
0006          DO 100 K=1,NLY          1161.000
0007          DO 100 M=1,18          1162.000
1
C
0008          FL=(STRSS(K,M,1)/XX)*2+(STRSS(K,M,3)/SS)*2          1163.000
0009          IF (FL .LT. FAL) GO TO 100          1164.000
0010          FAL=FL          1165.000
0011          KT=K          1166.000
0012          MT=M          1167.000
0013          IF (FAL .GT. 1.0) GO TO 200          1168.000
0014          100 CONTINUE          1169.000
1
C
0015          200 RETURN          1170.000
0016          END          1171.000
*OPTIONS IN EFFECT* ID,EBCDIC,SOURCE,NOLIST,NOECK,LOAD,NOMAP
*OPTIONS IN EFFECT* NAME = CFAIL          1172.000
*STATISTICS* SOURCE STATEMENTS = LINECNT = 57
*STATISTICS* PROGRAM SIZE = 16
*STATISTICS* NO DIAGNOSTICS GENERATED          1173.000
          584          1174.000
          1175.000
          1176.000
          1177.000
          1178.000
          1179.000

```

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8)

```

0001          SUBROUTINE  STRESS(DIS)
0002          IMPLICIT REAL*8(A-H,O-Z)
0003          C
0004          COMMON /ACONT/  ANG,DANG,NLY,ITEST,XX,SS,RT,RC,HI,ANT(9),THICK(9),N
1RAN
0005          COMMON /AMAT/  E1,E2,V12,G12,C(3,3),QA(40,3,3)
0006          COMMON /ALEM/  NODXY,NEGX,IJK(4,400)
0007          COMMON /ACOOR/  X(400),Y(400),RG(2),SG(2),WG(2)
1          SI(4),RI(4),P(20),Q(20)
0008          COMMON /AFUN/  SHP(3,4),T(40,3,3),STRSS(40,18,3),ST(11,3
1)          COMMON /ALOA/  XLC(20),YLC(20)
0009          C
0010          DIMENSION HP(3,4),LEB(20),XI(4),VI(4),DIS(2,400),STRAS(11,3)
0011          DO 10 I=1,18
0012          DO 10 J=1,3
0013          STRA(I,J)=O.O
10          CONTINUE
0014          DO 13 I=1,20
0015          XLC(1)=O.O
0016          13 YLC(1)=O.O
0017          C
0018          CALL POINT(LEB)
0019          DO 200 M=1,18
0020          II=LEB(M)
0021          II1=IJK(1,II)
0022          II2=IJK(2,II)
0023          II3=IJK(3,II)
0024          II4=IJK(4,II)
0025          X1(1)=X(II1)
0026          X1(2)=X(II2)
0027          X1(3)=X(II3)
0028          X1(4)=X(II4)
0029          C
0030          Y1(1)=Y(II1)
0031          Y1(2)=Y(II2)
0032          Y1(3)=Y(II3)
0033          Y1(4)=Y(II4)
0034          C
0035          C
0036          DO 100 N=1,4
0037          NN=IJK(N,II)
0038          STRA(M,1)=STRA(M,1)+SHP(1,N)*DIS(1,NN)
0039          STRA(M,2)=STRA(M,2)+SHP(2,N)*DIS(2,NN)

```

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8) STRESS 06-01-82 16:15:07 PAGE P002

```

0040      C      STRA(M,3)=STRA(M,3)+SHP(1,N)*DIS(2,NN)+SHP(2,N)*DIS(1,NN)
0041      C      100 CONTINUE
0042      C      DO 50 K=1,NLY
0043      C      DO 40 L=1,3
0044      C      STRSS(K,M,L)=O.O
0045      C      DO 30 I=1,3
0046      C      DO 20 J=1,3
0047      C      STRSS(K,M,L)=STRSS(K,M,L)+T(K,L,I)*QA(K,I,J)*STRA(M,J)
0048      C      20 CONTINUE
0049      C      30 CONTINUE
0050      C      40 CONTINUE
0051      C      50 CONTINUE
0052      C      200 CONTINUE
0053      C      250 DO 11 I=1,11
0054      C      DO 11 J=1,3
0055      C      11 STRAS(I,J)=O.O
0056      C      MX=148
0057      C      MY=154
0058      C      MT=O
0059      C      DO 400 IN=MX,MY
0060      C      MT=MT+1
0061      C      I11=IJK(1,IN)
0062      C      I12=IJK(2,IN)
0063      C      I13=IJK(3,IN)
0064      C      I14=IJK(4,IN)
0065      C      X1(1)=X(I11)
0066      C      X1(2)=X(I12)
0067      C      X1(3)=X(I13)
0068      C      X1(4)=X(I14)
0069      C      Y1(1)=Y(I11)
0070      C      Y1(2)=Y(I12)
0071      C      Y1(3)=Y(I13)
0072      C      Y1(4)=Y(I14)
0073      C      AP=RG(1)
0074      C      AQ=SG(2)
0075      C      CALL SHAPEF(IN,XSJ,AP,AQ)
0076      C      DO 500 NT=1,4
0077      C      NN=IJK(NT,IN)
0078      C      XLC(MT)=XLC(MT)+SHP(3,NT)*X(NN)
0079      C      YLC(MT)=YLC(MT)+SHP(3,NT)*Y(NN)
0080      C      STRAS(MT,1)=STRAS(MT,1)+SHP(1,NT)*DIS(1,NN)

```

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8)

	STRESS	06-01-82	16:15:07	PAGE 0003
0081	STRAS(MT,2)=STRAS(MT,2)+SHP(2,NT)*DIS(2,NN)	1288.000		
0082	STRAS(MT,3)=STRAS(MT,3)+SHP(1,NT)*DIS(2,NN)+SHP(2,NT)*DIS(1,NN)	1289.000		
0083	C 500 CONTINUE	1290.000		
0084	C DO 520 J=1,3	1291.000		
0085	520 ST(MT,J)=O,O	1292.000		
0086	DO 550 J=1,3	1293.000		
0087	ST(MT,1)=ST(MT,1)+C(1,J)*STRAS(MT,J)	1294.000		
0088	ST(MT,2)=ST(MT,2)+C(2,J)*STRAS(MT,J)	1295.000		
0089	ST(MT,3)=ST(MT,3)+C(3,J)*STRAS(MT,J)	1296.000		
0090	C 550 CONTINUE	1297.000		
0091	C C 400 CONTINUE	1298.000		
0092	C C MX=164	1299.000		
0093	MY=166	1300.000		
0094	MT=7	1301.000		
0095	DO 401 IN=MX,MY	1302.000		
0096	MT=MT+1	1303.000		
0097	I1=IJK(1,IN)	1304.000		
0098	I12=IJK(2,IN)	1305.000		
0099	I13=IJK(3,IN)	1306.000		
0100	I14=IJK(4,IN)	1307.000		
0101	C X1(1)=X(II1)	1308.000		
0102	X1(2)=X(II2)	1309.000		
0103	X1(3)=X(II3)	1310.000		
0104	X1(4)=X(II4)	1311.000		
0105	C Y1(1)=Y(II1)	1312.000		
0106	Y1(2)=Y(II2)	1313.000		
0107	Y1(3)=Y(II3)	1314.000		
0108	Y1(4)=Y(II4)	1315.000		
0109	C AP=RG(1)	1316.000		
0110	AQ=SG(2)	1317.000		
0111	C CALL SHAPEF(IN,XSJ,AP,AQ)	1318.000		
0112	C DO 501 NT=1,4	1319.000		
0113	NN=IJK(NT,IN)	1320.000		
0114	C XLC(MT)=XLC(MT)+SHP(3,NT)*X(NN)	1321.000		
0115	YLC(MT)=YLC(MT)+SHP(3,NT)*Y(NN)	1322.000		
0116	C STRAS(MT,1)=STRAS(MT,1)+SHP(1,NT)*DIS(1,NN)	1323.000		
0117	STRAS(MT,2)=STRAS(MT,2)+SHP(2,NT)*DIS(2,NN)	1324.000		
0118	STRAS(MT,3)=STRAS(MT,3)+SHP(1,NT)*DIS(2,NN)+SHP(2,NT)*DIS(1,NN)	1325.000		
0119	C 501 CONTINUE	1326.000		

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8)

STRESS 06-01-82 16: 15:07 PAGE P004

C 00 521 J=1,3
0120 521 ST(MT,J)=0.0
0121 DO 551 J=1,3
0122 ST(MT,1)=ST(MT,1)+C(1,J)*STRAS(MT,J)
0123 ST(MT,2)=ST(MT,2)+C(2,J)*STRAS(MT,J)
0124 ST(MT,3)=ST(MT,3)+C(3,J)*STRAS(MT,J)
0125
C 551 CONTINUE
0126
C
C 401 CONTINUE
CC DO 666 I=1,10
C WRITE(6,677) I,(STRAS(I,J),J=1,3)
C 677 FORMAT(15X,POINT=15.5X,'STRA11=',E15.8,
C 15X,'STRA12=',E15.8,/)
C 666 CONTINUE
C
0127 300 RETURN
0128 END
0129 *OPTIONS IN EFFECT* ID: EBCDIC, SOURCE, NOLIST, NODECK, LOAD, NOMAP
OPTIONS IN EFFECT NAME = STRESS LINECNT = 57
STATISTICS SOURCE STATEMENTS = 129, PROGRAM SIZE = 4434
STATISTICS NO DIAGNOSTICS GENERATED

AD-A121 407

STRENGTH OF MECHANICALLY FASTENED COMPOSITE JOINTS(U) 272

MICHIGAN UNIV ANN ARBOR DEPT OF MECHANICAL ENGINEERING

AND APPLIED MECHANICS F CHANG ET AL. JUL 82

UNCLASSIFIED

AFWAL-TR-82-4095 F33615-81-C-5050

F/G 13/5

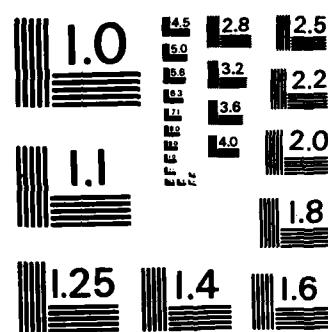
NL

END

FILED

1

0101



MICROCOPY RESOLUTION TEST CHART
NATIONAL BUREAU OF STANDARDS - 1963 - A

MICHIGAN TERMINAL SYSTEM FORTRAN G(21,8)

P/GE P002

1415.00
1416.00
1417.00
1418.00
1419.00
1420.00
1421.00
1422.00
1423.00
1424.00
1425.00
1426.00
1427.00
1428.00
1429.00
1430.00
1431.00
1432.00
1433.00
1434.00
1435.00
1436.00
1437.00
1438.00
1439.00
1440.00
1441.00
1442.00
1443.00
1444.00
1445.00
1446.00
1447.00
1448.00
1449.00
1450.00
1451.00
1452.00
1453.00
1454.00
1455.00
1456.00
1457.00
1458.00
1459.00
1460.00
1461.00
1462.00
1463.00
1464.00
1465.00

06-01-82 16:15:13

Michigan Terminal System FORTRAN G(21.8) OUTPUT

```

C 901 FORMAT (//,20X,'MATERIAL PROPERTIES OF SINGLE PLY')
311 FORMAT (/,20X,'E1=' ,F15.3,5X,'V12=' ,F10.3,
15X,'G12=' ,F15.3/)
213 FORMAT (/,20X,'** THE MAXIMUM ORIENTATION ANGLE=' ,
11X,F6.2,5X,'** THE INCREMENTAL ORIENTATION ANGLE=' ,
21X,F6.2/)
400 FORMAT (//,20X,'DIAMETER=' ,F10.4,5X,'W/D=' ,F10.4,/,20X,'E/D=' ,
1F10.4,5X,'L/D=' ,F10.4/,20X,'THICKNESS=' ,F10.5,/)
902 FORMAT (//,20X,'PLY ORIENTATION ' ,10X,'PLY THICKNESS' )
903 FORMAT (/,20X,'PLY' ,I3,'=' ,F7.3,11X,F8.5,5X)
904 FORMAT (//,20X,'GEOMETRY' )
905 FORMAT (/,20X,'/')

C      WRITE(6,101) PF
101 FORMAT(/,10X,***** THE MAXIMUM LOAD=' ,F10.3,2X,' *****,1
1X)
102 IF(MT .LE. 5) WRITE(6,91)
103 IF(MT .LE. 11 .AND. MT .GE. 8) WRITE(6,92)
104 IF(MT .LE. 14 .AND. MT .LE. 18) WRITE(6,93)
105 IF(MT .LE. 7 .AND. MT .GE. 6) WRITE(6,94)
106 IF(MT .GE. 12 .AND. MT .LE. 13 ) WRITE(6,95)
107 WRITE(6,102)
108 FORMAT(/,3X,'FAILURE LAYER' ,3X,'FAILURE POINT' ,
13X,'FAILURE POSITION' ,3X,'STRESS11' ,3X,'STRESS22' ,3X,'STRESS12' ,/)
109 WRITE(6,103) KT,MT,P(MT),Q(MT),SX1,SX52,SX53
110 FORMAT (/,7X,I3,12X,I3.7X,(' ,1X,F5.3,2X,F5.3,'),3X,E10.4,
13X,E10.4,3X,E10.4,/)
111
C      WRITE(6,96)
112 WRITE(6,97) TP
113 WRITE(6,98) FAL
114 WRITE(6,99)

C      91 FORMAT(//,10X,***** THE FAILURE MODE = BEARING MODE' //)
92 FORMAT(//,10X,***** THE FAILURE MODE = SHEAROUT MODE' //)
93 FORMAT(//,10X,***** THE FAILURE MODE = TENSION MODE' //)
94 FORMAT(//,10X,***** THE FAILURE MODE = BEARING AND SHEAROUT
1MODE' //)
95 FORMAT(//,10X,***** THE FAILURE MODE = TENSION AND SHEAROUT
1MODE' //)
96 FORMAT(//,10X,***** THE FAILURE MODE = BEARING MODE' //)
97 FORMAT(//,10X,***** THE INITIAL LOAD=' ,F15.5,/)
98 FORMAT(//,10X,***** THE FAILURE INDICTOR = ' ,F7.4,/)
99 FORMAT(//,10X,***** THE STRESS DISTRIBUTIONS DUE TO
1THE INITIAL LOAD ON THE CHARACTERISTIC CURVE ' ,//)
100 DO 200 K=1,NLY
101 WRITE(6,104)
102 FORMAT(//,3X,'LAYER' ,7X,'NO.' ,5X,X1,8X,'X2',8X,'XXX',10X,'YYY',10
1X,'TXY',/)
103 DO 150 N=1,18
104 WRITE(6,105) K,N,P(N),Q(N),STRSS(K,N,1),STRSS(K,N,2),STRSS(K,N,3)

```

```

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8)          OUTPUT          06-01-82    16:15:13    PAGE P003
0074      105 FORMAT(//,5X,13.5X,13.5X,F6.3,3X,F6.3,6X,E10.4,5X,E10.4,/) 1466.000
0075      150 CONTINUE
C
0076      200 CONTINUE
0077      WRITE(6,17)
0078      17 FORMAT(//,*****)
CACROSS THE LIGAMENT OF THE PLATE ***** THE STRESS DISTRIBUTIONS ****
C
0079      300 DO 301 I=1,10
0080      WRITE(6,305) I,XLC(I),YLC(I),(ST(I,J),J=1,3)
0081      305 FORMAT(//,5X,POINT='15.5X',X1='F7.4,5X',X2='F7.4,5X',T11='
1E15.8.5X',T22='E15.8.5X',T12='E15.8./')
0082      301 CONTINUE
C
0083      GO TO 500
0084      500 RETURN
0085      END
*OPTIONS IN EFFECT*  ID,EBCDIC,SOURCE,NOLIST,NOECK,LOAD,NOMAP
*OPTIONS IN EFFECT*  NAME = OUTPUT  LINECNT = 57
*STATISTICS*  SOURCE STATEMENTS = 85,PROGRAM SIZE = 4414
*STATISTICS*  NO DIAGNOSTICS GENERATED

```

1428
OPTIONS IN EFFECT 10. EBCDIC, SOURCE, NULLIS, NODECK, LUAD, NUMMAP
OPTIONS IN EFFECT NAME = FORMK LINECNT = 57
STATISTICS SOURCE STATEMENTS = 29, PROGRAM SIZE =
STATISTICS NO DIAGNOSTICS GENERATED
STATISTICS NO DIAGNOSTICS GENERATED

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8)

```

0001      SUBROUTINE BOUND
0002      IMPLICIT REAL*8 (A-H,O-Z)
0003      COMMON /ACONT/ ANG,DANG,NLY,ITEST,XX,SS,RT,RC,HI,ANT(9),THICK(9),N
1RAN
0004      COMMON /ADIM/ D,WD,ED,ALD,W,E,AL
0005      COMMON /ANGD/ NX,NP,NEO,NBC(37),NFIIX(37),LINT,NBAND
0006      COMMON /ALEM/ NODXY,NEGX,IJK(4,400)
0007      COMMON /AMAX/ SK(706,90),R1(706)
0008      C      NB=37
0009      C      IF(ITEST .EQ. 1) NB=33
0010      DO 500 N=1,NB
0011      NX=10
0012      I=NBC(N)
0013      NRQWB=(I-1)*2
0014      C      *** EXAMINE EACH DEGREE OF FREEDOM ****
0015      DO 490 M=1,2
0016      NRQWB=NRQWB+1
0017      ICON=NFIIX(N)/NX
0018      C      IF(ICON) 450,450,420
0019      DO 430 J=2,NBAND
0020      SK(NRQWB,J)=0.0
0021      NR=NRQWB+1-J
0022      C      IF(NR) 430,430,425
0023      C      425 SK(NR,J)=0.0
0024      C      430 CONTINUE
0025      C      NFIIX(N)=NFIIX(N)-NX*ICON
0026      C      450 NX=NX/10
0027      C      490 CONTINUE
0028      C      500 CONTINUE
0029      C      RETURN
0030      END
*OPTIONS IN EFFECT* ID,EBCDIC,SOURCE,NOLIST,NOECK,LOAD,NOMAP
*OPTIONS IN EFFECT* NAME = BOUND   LINECNT = 57
*STATISTICS* SOURCE STATEMENTS = 30,PROGRAM SIZE = 740
*STATISTICS* NO DIAGNOSTICS GENERATED

```

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8)

SOLVE 06-01-82 16: 15: 19 PAGE 001

```

0001      SUBROUTINE SOLVE
0002      IMPLICIT REAL*8 (A-H,O-Z)
0003      COMMON / AFORE / PF,DP,F(706)
0004      COMMON /ANQD/ NX,NP,NEQ,NBC(37),NFX(37),LINT,NBAND
0005      COMMON /ALEM/ NODXY,NELX,IJK(4,400)
0006      COMMON /AMAX/ SK(706,90),R1(706)

C      DO 100 I=1,NEQ
C      R1(I)=F(I)
C      100 CONTINUE

0010      DO 300 N=1,NEQ
I=N
0011      DO 290 L=2,NBAND
I=I+1

C      IF(I-NEQ) 230,230,290
C      230 IF(SK(N,L)) 240,290,240
C      240 C1=SK(N,L)/SK(N,1)
J=0
0014      DO 270 K=L,NBAND
J=J+1

C      IF(SK(N,K)) 260,270,260
C      260 SK(I,J)=SK(I,J)-C1*SK(N,K)
0015      270 CONTINUE

C      280 SK(N,L)=C1
R1(I)=R1(I)-C1*R1(N)
0016      290 CONTINUE

C      300 R1(N)=R1(N)/SK(N,1)
0017      310 CONTINUE

C      *** BACK SUBSTITUTION *****
C      N=NEQ
0018      350 N=N-1
C      IF(N) 500,500,360
0019      360 L=N
DO 400 K=2,NBAND
L=L+1

C      370 R1(N)=R1(N)-SK(N,K)*R1(L)
0020      400 CONTINUE
GO TO 350

```

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8)

0037 500 RETURN
0038 END
OPTIONS IN EFFECT 10 EBCDIC, SOURCE, NOLIST, NODECK, LOAD, NOMAP
OPTIONS IN EFFECT NAME = SOLVE LINECNT = 57
STATISTICS SOURCE STATEMENTS = 38, PROGRAM SIZE = 1068
STATISTICS NO DIAGNOSTICS GENERATED

06-01-82

16:15:19

PAGE 002

1622,000
1623,000

16:15:20

PAGE P001

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8)

		06-01-82	SHAPE
0001	SUBROUTINE SHAPEF(MM,XSJ,R,S)		1624.000
0002	IMPLICIT REAL*8 (A-H,O-Z)		1625.000
0003	COMMON /ANOD/ NX,NP,NEC(37),NFX(37),LINT,NBAND		1626.000
0004	COMMON /ALEM/ NODX,NELX,JK(4,400)		1627.000
0005	COMMON /ACODR/ X(400),Y(400),RG(2),SG(2),WG(2)		1628.000
1	COMMON /AFUN/ 'SI(4),RI(4),P(20),Q(20)		1629.000
0006	COMMON /AFUN/ SHP(3,4),T(40,3,3),STRSS(40,18,3),ST(11,3)		1630.000
1			1630.000
0007	C		1631.000
	C		1632.000
	C		1633.000
	C		1634.000
	C		1635.000
	C		1636.000
	C		1637.000
	C		1638.000
	C		1639.000
0008	DO 100 I=1,NP		1640.000
0009	C		1641.000
0010	C		1642.000
0011	C		1643.000
0012	C		1644.000
	C		1645.000
	C		1646.000
	C		1647.000
	C		1648.000
	C		1649.000
0013	DO 200 I=1,2		1650.000
0014	DO 200 J=1,2		1651.000
0015	200 XS(I,J)=0.0		1652.000
	C		1653.000
	C		1654.000
	C		1655.000
	C		1656.000
	C		1657.000
	C		1658.000
	C		1659.000
0016	DO 210 J=1,2		1660.000
0017	DO 210 K=1,NP		1661.000
0018	NN=IJK(K,MM)		1662.000
0019	XS(1,J)=XS(1,J)+X(NN)*SHP(J,K)		1663.000
0020	XS(2,J)=XS(2,J)+Y(NN)*SHP(J,K)		1664.000
0021	C		1665.000
	C		1666.000
	C		1667.000
	C		1668.000
	C		1669.000
0022	C		1670.000
	C		1671.000
	C		1672.000
	C		1673.000
0023	DO 300 I=1,4		1674.000
0024	C		1675.000
0025	C		1676.000
0026	C		1677.000
	C		1678.000
	C		1679.000
	C		1680.000
	C		1681.000
	C		1682.000
	C		1683.000
	C		1684.000
	C		1685.000
	C		1686.000
	C		1687.000
	C		1688.000
	C		1689.000
	C		1690.000
	C		1691.000
	C		1692.000
	C		1693.000
	C		1694.000
	C		1695.000
	C		1696.000
	C		1697.000
	C		1698.000
	C		1699.000
	C		1700.000
	C		1701.000
	C		1702.000
	C		1703.000
	C		1704.000
	C		1705.000
	C		1706.000
	C		1707.000
	C		1708.000
	C		1709.000
	C		1710.000
	C		1711.000
	C		1712.000
	C		1713.000
	C		1714.000
	C		1715.000
	C		1716.000
	C		1717.000
	C		1718.000
	C		1719.000
	C		1720.000
	C		1721.000
	C		1722.000
	C		1723.000
	C		1724.000
	C		1725.000
	C		1726.000
	C		1727.000
	C		1728.000
	C		1729.000
	C		1730.000
	C		1731.000
	C		1732.000
	C		1733.000
	C		1734.000
	C		1735.000
	C		1736.000
	C		1737.000
	C		1738.000
	C		1739.000
	C		1740.000
	C		1741.000
	C		1742.000
	C		1743.000
	C		1744.000
	C		1745.000
	C		1746.000
	C		1747.000
	C		1748.000
	C		1749.000
	C		1750.000
	C		1751.000
	C		1752.000
	C		1753.000
	C		1754.000
	C		1755.000
	C		1756.000
	C		1757.000
	C		1758.000
	C		1759.000
	C		1760.000
	C		1761.000
	C		1762.000
	C		1763.000
	C		1764.000
	C		1765.000
	C		1766.000
	C		1767.000
	C		1768.000
	C		1769.000
	C		1770.000
	C		1771.000
	C		1772.000
	C		1773.000
	C		1774.000
	C		1775.000
	C		1776.000
	C		1777.000
	C		1778.000
	C		1779.000
	C		1780.000
	C		1781.000
	C		1782.000
	C		1783.000
	C		1784.000
	C		1785.000
	C		1786.000
	C		1787.000
	C		1788.000
	C		1789.000
	C		1790.000
	C		1791.000
	C		1792.000
	C		1793.000
	C		1794.000
	C		1795.000
	C		1796.000
	C		1797.000
	C		1798.000
	C		1799.000
	C		1800.000
	C		1801.000
	C		1802.000
	C		1803.000
	C		1804.000
	C		1805.000
	C		1806.000
	C		1807.000
	C		1808.000
	C		1809.000
	C		1810.000
	C		1811.000
	C		1812.000
	C		1813.000
	C		1814.000
	C		1815.000
	C		1816.000
	C		1817.000
	C		1818.000
	C		1819.000
	C		1820.000
	C		1821.000
	C		1822.000
	C		1823.000
	C		1824.000
	C		1825.000
	C		1826.000
	C		1827.000
	C		1828.000
	C		1829.000
	C		1830.000
	C		1831.000
	C		1832.000
	C		1833.000
	C		1834.000
	C		1835.000
	C		1836.000
	C		1837.000
	C		1838.000
	C		1839.000
	C		1840.000
	C		1841.000
	C		1842.000
	C		1843.000
	C		1844.000
	C		1845.000
	C		1846.000
	C		1847.000
	C		1848.000
	C		1849.000
	C		1850.000
	C		1851.000
	C		1852.000
	C		1853.000
	C		1854.000
	C		1855.000
	C		1856.000
	C		1857.000
	C		1858.000
	C		1859.000
	C		1860.000
	C		1861.000
	C		1862.000
	C		1863.000
	C		1864.000
	C		1865.000
	C		1866.000
	C		1867.000
	C		1868.000
	C		1869.000
	C		1870.000
	C		1871.000
	C		1872.000
	C		1873.000
	C		1874.000
	C		1875.000
	C		1876.000
	C		1877.000

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8)

OPTIONS IN EFFECT ID.EBCDIC, SOURCE.NOLIST, NODECK, LOAD, NOMAP
OPTIONS IN EFFECT NAME = SHAPEF LINECNT * 57
STATISTICS SOURCE STATEMENTS = 28, PROGRAM SIZE = 1028
STATISTICS NO DIAGNOSTICS GENERATED

SHAPEF 06-01-82 16:15:20 PAGE E002

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8)

```

0001      SUBROUTINE NEWTON(IN,AP,AQ,R,S)          06-01-82   16:15:21   PAGE: P001
0002      IMPLICIT REAL*8 (A-H,O-Z)               1678.000
0003      COMMON /ACCDR/ X(400),Y(400),RG(2),SG(2),WG(2)
1           SI(4),RI(4),P(20),Q(20)             1679.000
0004      COMMON /ANOD/ NX,NP,NEQ,NBC(37),NFIIX(37),LINT,NBAND
0005      COMMON /ALEM/ NODXY,NELX,IJK(4,400)
0006      DIMENSION RP(2)                         1680.000
0007      F(R,S)=A3*R+S+A2*S+A1*R+AO-AP      1681.000
0008      G(R,S)=B3=R*S+B2*S+B1*R+BO-AQ      1682.000
0009      C           FR(R,S)=A3*S+A           1683.000
0010      C           FS(R,S)=A3*R+A2          1684.000
0011      C           GR(R,S)=B3*S+B1          1685.000
0012      C           GS(R,S)=B3*R+B2          1686.000
0013      C           AO=0.0                  1687.000
0014      C           A1=0.0                  1688.000
0015      C           A2=0.0                  1689.000
0016      C           A3=0.0                  1690.000
0017      C           BO=0.0                  1691.000
0018      C           B1=0.0                  1692.000
0019      C           B2=0.0                  1693.000
0020      C           B3=0.0                  1694.000
0021      C           DO 100 I=1,4          1695.000
0022      C           N=IJK(I,IN)           1696.000
0023      C           AO=AO+0.25*X(N)          1697.000
0024      C           A1=A1+0.25*X(N)*RI(I)      1698.000
0025      C           A2=A2+0.25*X(N)*SI(I)      1699.000
0026      C           A3=A3+0.25*X(N)*RI(I)*SI(I)  1700.000
0027      C           BO=BO+0.25*Y(N)          1701.000
0028      C           B1=B1+0.25*Y(N)*RI(I)      1702.000
0029      C           B2=B2+0.25*Y(N)*SI(I)      1703.000
0030      C           B3=B3+0.25*Y(N)*RI(I)*SI(I)  1704.000
0031      C           100 CONTINUE          1705.000
0032      C           R=0.0                  1706.000
0033      C           S=0.0                  1707.000
0034      C           AA=F(R,S)           1708.000
0035      C           BB=G(R,S)           1709.000
0036      C           WRITE(6,3) R,S,AA,BB      1710.000
0037      C           DO 10 I=1,20          1711.000
0038      C           XJ=FR(R,S)*GS(R,S)-FS(R,S)*GR(R,S)  1712.000
0039      C           IF(XJ.EQ.0.0) WRITE(6,6)          1713.000
0040      C           IF(XJ.EQ.0.0) GO TO 150          1714.000
0041      C           100 CONTINUE          1715.000
0042      C           R=0.0                  1716.000
0043      C           S=0.0                  1717.000
0044      C           100 CONTINUE          1718.000
0045      C           R=0.0                  1719.000
0046      C           S=0.0                  1720.000
0047      C           AA=F(R,S)           1721.000
0048      C           BB=G(R,S)           1722.000
0049      C           WRITE(6,3) R,S,AA,BB      1723.000
0050      C           DO 10 I=1,20          1724.000
0051      C           XJ=FR(R,S)*GS(R,S)-FS(R,S)*GR(R,S)  1725.000
0052      C           IF(XJ.EQ.0.0) WRITE(6,6)          1726.000
0053      C           IF(XJ.EQ.0.0) GO TO 150          1727.000
0054      C           100 CONTINUE          1728.000
0055      C           R=0.0                  1729.000
0056      C           S=0.0                  1730.000
0057      C           AA=F(R,S)           1731.000
0058      C           BB=G(R,S)           1732.000

```

MICHIGAN TERMINAL SYSTEM FORTRAN G(21.8)

```

      NEWTON          06-01-82      16: 15:21      PAGE P002

0040      C      DELTAX=(-F(R,S)*GS(R,S)+G(R,S)*FS(R,S))/XJ      1733.000
0041      C      DELTAY=(-G(R,S)*FR(R,S)+F(R,S)*GR(R,S))/XJ      1734.000
0042      C      R=R+DELTAX      1735.000
0043      C      S=S+DELTAY      1736.000
0044      C      AA=F(R,S)      1737.000
0045      C      BB=G(R,S)      1738.000
0046      C      WRITE(6,4) I,R,S,AA,BB      1739.000
      C      IF(DABS(DELTAX) .LT. 1.E-7 .AND. DABS(DELTAY) .LT. 1.E-7)      1740.000
      C      1GO TO 150      1741.000
0047      C      10 CONTINUE      1742.000
0048      C      WRITE(6,5)
      C      3 FORMAT(//,10X,'R',15X,'S',15X,'F(R,S)',15X,'G(R,S)',/)
      C      15X,F10.7,5X,F12.7,5X,F12.7/)
      C      4 FORMAT(//,15X,F10.7,5X,F10.7,5X,F12.7,5X,F12.7/)
      C      5 FORMAT(//,13,F10.7,5X,F10.7,5X,F12.7,5X,F12.7/)
0049      C      6 FORMAT(//,'FAIL TO CONVERGE IN 20 ITERATION ',/)
0050      C      6 FORMAT(//,'*** JOCOBIAN=0.0 ***',/)

0051      C      150 RETURN      1743.000
0052      C      END
      *OPTIONS IN EFFECT* ID,EBCDIC,SOURCE,NOLIST,NODECK,LOAD,NOMAP      1744.000
      *OPTIONS IN EFFECT* NAME = NEWTON      57
      *STATISTICS* SOURCE STATEMENTS = 52,PROGRAM SIZE = 2348
      *STATISTICS* NO DIAGNOSTICS GENERATED      1745.000
      1746.000
      1747.000
      1748.000
      1749.000
      1750.000
      1751.000
      1752.000
      1753.000
      1754.000
      1755.000
      1756.000

```

```

      *OPTIONS IN EFFECT* ID,EBCDIC,SOURCE,NOLIST,NODECK,LOAD,NOMAP
      *OPTIONS IN EFFECT* NAME = NEWTON      57
      *STATISTICS* SOURCE STATEMENTS = 52,PROGRAM SIZE = 2348
      *STATISTICS* NO DIAGNOSTICS GENERATED

```

NO STATEMENTS FLAGGED IN THE ABOVE COMPILATIONS.

C TEST SAMPLE

CONSIDER A SINGLE FASTENER COMPOSITE JOINT MADE OF T300/SP286 GRAPHITE EPOXY LAMINATE. THE MATERIAL PROPERTIES ARE GIVEN AS E1=18.70 E06 PSI, G12=0.719 E06 PSI, V12=0.30, X(PLY TENSILE STRENGTH)=0.15 E 06 PSI, SS(LAMINATE SHEAR STRENGTH--CROSS PLY)=0.018 E06 PSI. QUASI-ISOTROPIC LAMINATE IS USED IN THIS CALCULATIONS.

THE GEOMETRY OF THE JOINT ARE DESCRIBED AS BELOW:

D(DIAMETER)=0.25 IN
W/D(WIDTH RATIO)=6.0
E/D(EDGE RATIO)=3.0
L/D(LENGTH RATIO)=14.0
H(THICKNESS)=0.021 IN
THE CHARACTERISTIC LENGTH FOR TENSION IS EQUAL TO 0.043 IN AND FOR COMPRESSION IS 0.120 IN.

THE FORMAT OF INPUT DATA IS GIVEN IN THE FOLLOWING:
(SEE INPUT INSTRUCTIONS IN PROGRAM)

1	0.0	1.0	719000.0	14.0
1870000.0	0.30	3.000	0.043	0.1200
0.2500	6.000			
150000.	18000.0			
0.021				
4	0.0	0.00525	45.0	0.00525
		0.00525	90.0	0.00525

Ou'put

***** THE STRENGTH PREDICTION OF FASTENED COMPOSITE JOINTS *****

MATERIAL PROPERTIES OF SINGLE PLY
 E1= 1870000.000 V12= 0.300 G12= 719000.000
 MATERIAL PROPERTIES
 X(PLY T-STRENGTH)= 150000.000 SC(LAMINATE S-STRENGTH)= 18000.000
 RT(CHAR - TEN.)= 0.0430 RC(CHAR - COMP.)= 0.1200
 PLY ORIENTATION PLY THICKNESS
 PLY 1= 0.0 0.00525
 PLY 2= 45.000 0.00525
 PLY 3=-45.000 0.00525
 PLY 4= 90.000 0.00525
 GEOMETRY
 DIAMETER= 0.2500 W/D= 6.0000
 E/D= 3.00000 L/D= 14.00000
 THICKNESS= 0.04200

***** THE MAXIMUM LOAD= 1161.505 *****
 ***** THE FAILURE MODE = BEARING AND SHEAROUT MODE *****
 FAILURE LAYER FAILURE POINT FAILURE POSITION STRESS11 STRESS22 STRESS12
 2 6 (0.102 0.215) -1.1448E+06 0.1820E+00 -.4687E+04

***** THE INITIAL LOAD= 42.00000 *****
 ***** THE FAILURE INDICATOR = 0.0013 *****
 ***** THE STRESS DISTRIBUTIONS DUE TO THE INITIAL LOAD ON THE CHARACTERISTIC CURVE FOR EACH PLY *****

LAYER	NO.	X1	X2	TXX	TYY	TXY
1	1	0.004	0.245	-1.4941E+04	0.8567E-02	0.3624E+02
1	2	0.012	0.245	-1.4906E+04	0.8634E-02	0.4183E+02
1	3	0.036	0.242	-1.4525E+04	0.7968E-02	0.1229E+03
1	4	0.059	0.235	-1.3780E+04	0.6694E-02	0.1953E+03
1	5	0.081	0.226	-1.2723E+04	0.4975E-02	0.2530E+03
1	6	0.102	0.215	-1.3422E+04	-2.146E-03	0.3091E+03
1	7	0.120	0.201	-1.1855E+04	-2.014E-02	0.3311E+03
1	8	0.137	0.185	-1.1380E+03	-3.547E-02	0.3319E+03
1	9	0.151	0.167	0.1630E+04	-4.515E-02	0.3095E+03
1	10	0.163	0.148	0.2002E+04	-6.198E-02	0.2530E+03
1	11	0.172	0.127	0.2841E+04	-7.092E-02	0.2235E+03
1	12	0.178	0.107	0.3552E+04	-6.921E-02	0.1816E+03
1	13	0.182	0.086	0.4095E+04	-5.661E-02	0.1340E+03
1	14	0.183	0.065	0.4461E+04	-3.454E-02	0.8795E+02
1	15	0.181	0.045	0.4572E+04	-3.443E-02	0.1065E+03
1	16	0.177	0.026	0.4909E+04	0.2985E-03	0.6738E+02
1	17	0.172	0.008	0.5220E+04	0.3879E-02	0.4397E+02
1	18	0.169	0.003	0.5407E+04	0.5019E-02	0.2622E+02
LAYER	NO.	X1	X2	TXX	TYY	TXY
2	1	0.004	0.245	-1.400E+04	-4.691E-02	-3.086E+03
2	2	0.012	0.245	-1.454E+04	-4.289E-02	-3.073E+03
2	3	0.036	0.242	-1.2437E+04	0.1522E-03	-2.835E+03
2	4	0.059	0.235	-1.3237E+04	0.4662E-02	-2.371E+03
2	5	0.081	0.226	-1.3777E+04	0.8923E-02	-1.719E+03
2	6	0.102	0.215	-1.5237E+04	0.6581E-02	-1.695E+03
2	7	0.120	0.201	-1.5151E+04	0.1032E-01	-7.824E+02
2	8	0.137	0.185	-1.4696E+04	0.1352E-01	0.1860E+02

POINT		1	X1 = 0.1271	X2 = 0.0026	111 = 0.10461911E+03	T22 = 0.43780118F+04	T12 = 0.10341868E+02
POINT		2	X1 = 0.1397	X2 = 0.0029	111 = 0.29470390E+03	T22 = 0.33064404E+C4	T12 = -0.26063698E+02
POINT		3	X1 = 0.1627	X2 = 0.0034	111 = 0.34292025E+03	T22 = 0.22448220E+04	T12 = -0.79359774E+02
POINT		4	X1 = 0.1962	X2 = 0.0041	111 = 0.29215934E+03	T22 = 0.15173410E+04	T12 = -0.11360285E+03
POINT		5	X1 = 0.2401	X2 = 0.0050	111 = 0.21404444E+03	T22 = 0.10787318E+04	T12 = -0.12442130E+03
POINT		6	X1 = 0.2944	X2 = 0.0061	111 = 0.14530340E+03	T22 = 0.81755422E+03	T12 = -0.11835797E+03
POINT		7	X1 = 0.3591	X2 = 0.0075	111 = 0.93844418E+02	T22 = 0.65169154E+03	T12 = -0.10077351E+03
POINT		8	X1 = 0.4401	X2 = 0.0091	111 = 0.55440756E+02	T22 = 0.53468656E+03	T12 = -0.71020255E+02
POINT		9	X1 = 0.5513	X2 = 0.0091	111 = 0.25849230E+02	T22 = 0.42057408E+03	T12 = -0.36908192E+02